

# Palestine Polytechnic University



College of Engineering & Technology  
Mechanical Engineering Department

Graduation Project

The design of mechanical systems in residential Villa  
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## Abstract

This project comes as the fruit of five years of hard work in the field of Refrigeration and Air conditioning engineering.

In this study a general study will be proposed for a villa in Ram Allah to design the the central heating, hot & cold water, sanitary drainage with high efficiency and good design.

Our project will contains soft ware discus these problems and contains presentation to expose our study to the doctors and discus them after we will finish from this project.

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## Introduction

### 1.1 Introduction

This project is a part of the course 'Introduction to Mechanical Design' at the University of... The project is to design a... for a... machine.

## CHAPTER ONE

## INTRODUCTION

The purpose of this project is to design a... for a... machine. The design is to be done using... The design is to be done using... The design is to be done using...

The design is to be done using... The design is to be done using... The design is to be done using...

The design is to be done using... The design is to be done using... The design is to be done using...

### 1.2 Project Objectives

1. To design a... for a... machine.
2. To produce the... drawings for the... machine.
3. To select the... materials and... for the... machine.
4. To produce the... drawings for the... machine.



# 1 Introduction

## 1-1 Introduction

In this project, a trial is made to make the best Central Heating System to cover heating requirements in a Villa in Ram Allah

A central heating system consists of all the pipe work and radiators that are connected to the boiler. The boiler provides the heat but, it's the pump (Circulator) that moves the heated water from the boiler through the pipe work to the radiators, and back to the boiler for re-heating. There are many types of systems that can be installed, and which may be tailored to your own preferences. But a carefully designed and installed system will give many years of trouble free running and will not waste heat, therefore keeping fuel costs low.

The system with the least cost and full ability to cover the whole energy demand is designed with all the accessories required. Separate drawings are including in the project with complete specifications. A complete bill is given in local prices. At last, a discussion and recommendations paper is included.

So, central heating system, sanitary system and water supply system are represented on the drawings

## 1-2 Project Objectives

- 1- Our project is to make design for central heating system and plumbing system
- 2- To prepare the required drawings for the above mentioned systems
- 3- To select the required equipments and parts and its specification of boiler, pipes... etc
- 4- Implementing the mechanical systems after a full study done

### 1-3 Project Benefits

We choose this project to prepare a full document for mechanical systems in the villa.

So, documents include

- 1- Heating load calculation for central heating system
- 2- Calculation for flow and pipes for plumping system
- 3- prepare the required drawings for the above mentioned systems
- 4- selection for the required equipments and parts and its specification of boiler, pipes... etc
- 5- Implementing the mechanical systems on the site

### 1-4 Budget

TASK	COST (NIC)
USING INTERNET	600
TRANSPORTATION FROM AND INTO THE PROJECT	200
OTHER TRANSPORTATION	100
PRINTING PAPERS	200
PRINTING DRAWING	600
TOTAL	1700

Table 1-1 Budgets

## 1-5 Time Planning

The project plan follows the following time schedule, which includes the related tasks of study and system analysis.

The following time plan is through the semester

### The Semester Time Plan

Task/Week	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16
Collecting Information about the project	█	█	█	█	█											
Study the mechanical draw.			█	█	█	█	█									
central heating system calculation & plumping system calculation						█	█	█	█							
Drawing planes & Selection of equipments							█	█	█	█	█					
Making bill of quantity (BOQ)									█	█	█	█	█	█		
Project Documentation	█	█	█	█	█	█	█	█	█	█	█	█	█	█	█	█

Fig 1.1 the semester Time Plan

## 2.1 INTRODUCTION

This chapter will discuss the components of a central heating system and how they are connected to the boiler. The boiler provides the heat for the pump, which circulates the water around the house. The water then flows through the radiators, which heat the room. The water then flows back to the boiler to be heated again.

A central heating system consists of all the pipes, radiators, and boiler connected to the boiler. The boiler provides the heat for the pump, which circulates the water around the house. The water then flows through the radiators, which heat the room. The water then flows back to the boiler to be heated again.

# CHAPTER TWO

## CENTRAL HEATING SYSTEM COMPONENTS



Fig. 2.1 Central heating system components

## 2.1 INTRODUCTION

This chapter deals with the components used in central heating system and talks about the role of each component in the system

A central heating system consists of all the pipe work and radiators that are connected to the boiler. The boiler provides the heat but, it's the pump (Circulator) that moves the heated water from the boiler through the pipe work to the radiators, and back to the boiler for re-heating. There are many types of systems that can be installed, and which may be tailored to your own preferences. But a carefully designed and installed system will give many years of trouble free running and will not waste heat, therefore keeping fuel costs low.

### Hot Water (Hydronic) Heating System

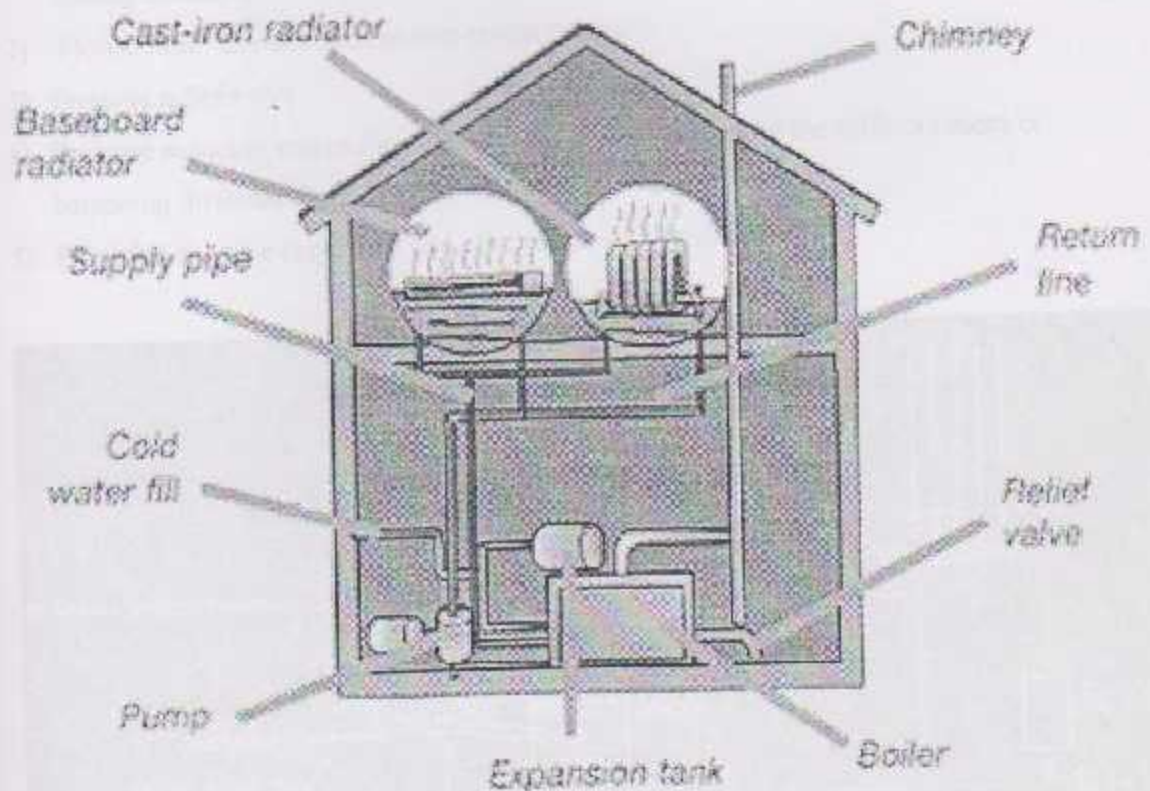


Fig 2.1 Central heating system components

## 2.2 Hot water heating system components

The main components of a central heating system using water circulation are:

- 1) Room heating elements (radiators or fan coils).
- 2) Boilers.
- 3) Piping network.
- 4) Circulating pump.
- 5) Expansion tank.
- 6) Oil tank.
- 7) Chimney.

Device and fitting are required in a hot water heating system, these include the following:

- 1) Air elimination device such as automatic air vent and manual air vents for room heating elements.
- 2) Flow control devices such as non-return valves.
- 3) Pressure relief valve.
- 4) Pressure reducing valves for balancing the piping network of the different room or balancing different zones.
- 5) Provision for pipe expansion and elimination of vibration.

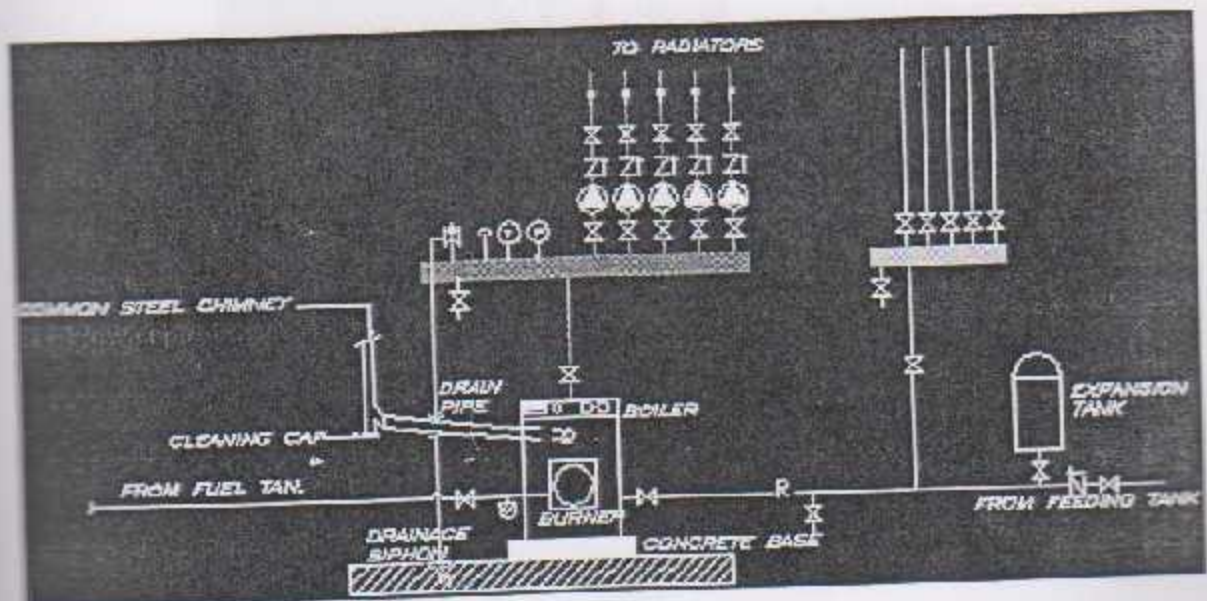


Fig 2.2 Central heating system

### 2.2.1 Room heating elements

Heat transfer elements facilitate the transfer of heat between the hot water and space to be heated; heating elements that are in common use in hot water heating system are Radiators

They are made of:-

- 1- Cast iron Radiators
- 2- Aluminum Radiators
- 3- Steel plates Radiators

So, we will use Aluminum Radiators in our project because it has low cost, light weight construction near 1.5 kg for one section , Limited thermal inertia and they are made of sections so could be lengthened and shortened easily

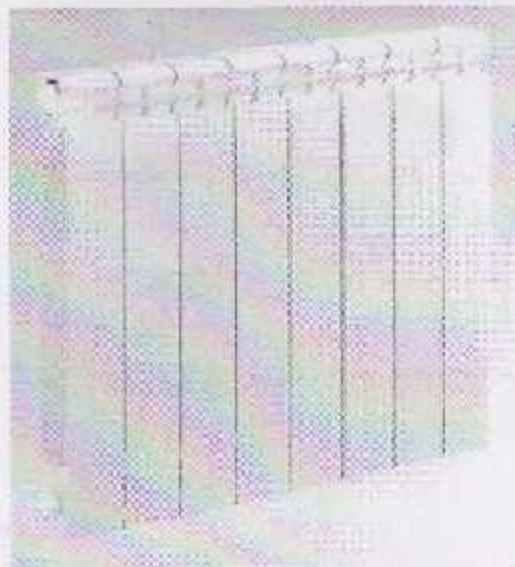


Fig 2.3 Aluminum radiator

### 2.2.2 Boilers

Boilers are constructed of cast iron or steel sheets in various capacities and sizes, they may be low pressure "200 kpa" or high pressure boilers "3200 kpa".

Boilers are classified in a number of ways as follows:-

- a) They may be classified on the basis of material out of which they are made as cast iron boilers or steel sheet boilers
- b) They may be classified on the basis of the operation pressure for example low pressure or high pressure boilers.
- c) They may be also classified on the basis of raw energy they use, for example, gas boiler, diesel boilers, or any other energy boiler.
- d) Type of construction may be used for classification for example, fire tube or water tube boilers.

The boiler package contains, in addition to the boiler body, the burner, the safety control and jacket, the safety devices associated with boilers include relief valves, water level cut-off device, switch on-off device, and outlet and inlet temperature control thermostat.



Fig 2.4 Burner

### 2.2.3 Piping network

We use radial pipe system, in this pipe system uses plastic pipe instead of metal pipe for circulating the hot water throughout the heating system.

The supply pipes of a certain number of radiators are connected to a single supply manifold, using special brass coupling, the return pipes are also connected to single return manifold. The supply and return manifold are respectively connected to the main supply and main return pipes of boiler which may be made from plastic or metal pipes.



### 2.2.4 Circulating pump

A circulating pump is simply a small pump whose purpose is to circulate the water in the hydraulic heating system; centrifugal pumps are the type normally used as circulators.

Modern hydronic heating systems are closed therefore, there is no static head for the pump to overcome since the weight of water in riser piping is exactly counter balanced by an equal weight of water in the return piping, the circulator, therefore must overcome only the friction head in the piping, the circulator is therefore usually driven by small, frictional, horse power electrical motors.

Circulator pumps are normally low head, low flow units, they are low head because of absence of static head, and low flow, because of the high heat carrying capacity of water, they are either installed in line when the water flow from pump inlet to outlet is in the same direction or as base mounted where a right angle change in water direction from inlet to outlet is desired, most circulators are installed in line.

The circulator can be installed either in the main supply line, outgoing from the boiler, or in the main return line coming back to the boiler, in systems with high head, it is normal practice to place the pump so that it pumps away from the boiler and expansion tank.



Fig 2.5 pump

### 2.2.5 Expansion tank

In a closed recirculating system (central heating system) a given amount of air or gas space is required to accommodate water expansion and pressurization. Since water expands when heated and contracts when cooled in direct proportion to the temperature change, and since water is incompressible, a lack of expansion space means that any volume increase due to heating will cause an immediate and definite pressure increase. Expansion tank is the primary device used to accommodate

fluctuations in water volume within a closed system while maintaining a predetermined range of pressures, from the minimum cold-fill pressure up to (but not more than) the maximum working pressure of the system.



Fig 2.6 Close expansion tank

#### 2.2.6 Oil tank.

A fuel storage tank can be an underground storage tank or an above ground storage tank as needed in the project.

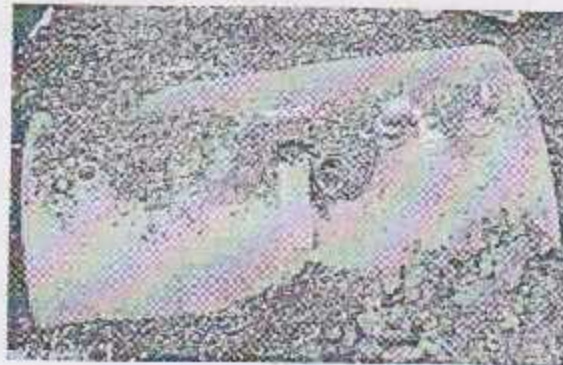


Fig 2.7 Underground oil tank

#### 2.2.7 Chimney

A vertical hollow structure of steel, or reinforced concrete, built to convey gaseous products of combustion from a building or process facility. A chimney should be high enough to furnish adequate draft and to discharge the products of combustion without causing local air pollution.

### 2.2.7.1 Types of chimney:

- 1- Cast-in-place concrete
- 2- Stainless steel.
- 3- Galvanized or aluminum sleeves

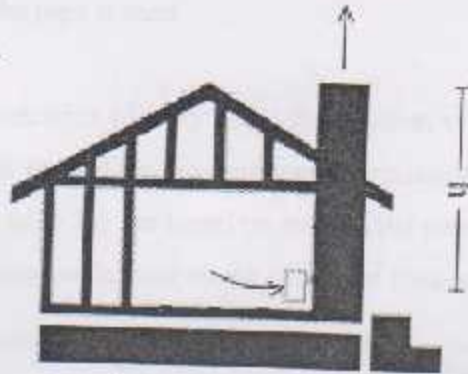


Fig 2.8 Chimney

## 2.8 Piping design

This section represents the principles and currently accepted design techniques for water piping systems used in air-conditioning applications. The principles and techniques described are applicable to hot water system. The water system discussed here is a recirculation type where the water is not discharged but flows in a repeating circuit from the heat exchanger to the heat exchanger. It is also a closed system because the flow of water is not exposed to the atmosphere at any point except at an insignificant area in the expansion tank.

### 2.8.1 Water piping design

#### 2.8.1.1 Pipe friction loss

The pipe friction loss in a system depends on water velocity, pipe diameter, interior surface roughness and pipe length. Fig.2-1 (See Appendix A) shows the friction chart for closed recirculation piping system. The chart shows the water velocity, pipe or tube diameter and water quantity in addition to the friction rate per 100ft of equivalent pipe length.

### 2.8.1.2 Water velocity

The values of velocities recommended for water piping depend on two conditions:

- 1- The service for which the pipe is used.
- 2- The effects of erosion.

Table 2-1 lists recommended velocity range for different services. The design of water of piping systems is limited by the maximum permissible flow velocity. The maximum values listed in table 2-1 are based on established permissible sound levels of moving water and entrained water and on the effects of erosion.

service	Velocity range(FPM)
Pump discharge	4-10
Pump suction	4-7
Drain line	3-10
header	5-15
riser	5-15
General service	5-12

Table 2-1 shows values of recommended water velocities (FPM)

### 2.8.1.3 Pipe Equivalent length

The total equivalent length of a water piping system is determined from the straight length in addition of equivalent length of pipes due to fittings, valves and other elements in the piping system. Tables 2-6, 2-7 and 2-8 (See Appendix A) give the additional equivalent lengths for pipe for these various components.

## 2.9 Water Cycle in Central Heating System

A central heating system consists of all the pipe work and radiators that are connected to the boiler. The boiler provides the heat but, it's the pump (Circulator) that moves the heated water from the boiler through the pipe work to the radiators, and back to the boiler for re-heating. There are many types of systems that can be installed, and which may be tailored to your own preferences. But a carefully designed and installed system will give many years of trouble free running and will not waste heat, therefore keeping fuel costs low.

## XI INTRODUCTION

The objective of this book is to provide a comprehensive guide to the design and selection of heating systems for buildings.

The book is divided into three main parts. The first part deals with the fundamentals of heat transfer and the calculation of heat loads. The second part deals with the selection of heating systems and the design of the distribution system. The third part deals with the control and operation of heating systems.

The book is intended for use by students of mechanical engineering and by engineers and architects who are concerned with the design and selection of heating systems for buildings.

# CHAPTER THREE

## HEAT LOAD CALCULATION & HEATING SYSTEM SELECTION

Heat loads are generally calculated from the following data:

1. Dimensions of the room or building and the temperature difference between the room and the outside air.
2. Thermal conductivity of the walls, ceiling, floor and windows.
3. Temperature of the room and the outside air.
4. Heat transfer coefficient of the walls, ceiling, floor and windows.

### 3.1 INTRODUCTION

This chapter deals with the usual procedures used to estimate heating loads, in order to size heating equipments.

Heat transfer through a building envelope is influenced by the materials used; by geometric factors such as size, shape, and orientation; by the existence of internal heat sources; and by climatic factors.

The primary function of heat loss calculations is to estimate the capacity that will be required for various heating components necessary to maintain comfort within a space.

These calculations are therefore based on peak load calculations for heating and correspond to environmental conditions which are near the extremes normally encountered.

**Loads are generally divided into the following two categories:**

- 1- Transmission: heat loss or heat gain due to a temperature difference across a building element.
- 2- Infiltration: heat loss due to the infiltration of outside air into a conditioned space.
- 3- Ventilation: heat loss due to the ventilation of outside air into a conditioned space

### 3.2 GENERAL PROCEDURE

The following procedure is used to estimate the heating load for a space. The various quantities can be influenced by factors:

- 1- Design winter air conditions
- 2- Indoor design air conditions based on the design conditions specified in the code
- 3- The temperature in the unconditioned adjacent space is estimated. Usually the average temperature between outdoor and indoor air is used
- 4- Heat transfer coefficients are determined for the walls, floor, ceiling, and windows from the code specifications. Surface between unconditioned space and conditioned space should be treated

### 3.2 Symbols

QH	heating load (W)
U (wall)	overall heat transfer coefficient for external wall ( $W/m^2 \cdot ^\circ C$ )
U (roof)	overall heat transfer coefficient for roof ( $W/m^2 \cdot ^\circ C$ )
U (ceiling)	overall heat transfer coefficient for ceiling ( $W/m^2 \cdot ^\circ C$ )
U (floor)	overall heat transfer coefficient for floor ( $W/m^2 \cdot ^\circ C$ )
U (partition)	overall heat transfer coefficient for internal partition ( $W/m^2 \cdot ^\circ C$ )
A	area ( $m^2$ )
TD	temperature difference ( $^\circ C$ )
HRD	humidity ratio difference (Kg H <sub>2</sub> O/Kg dry air)
NO.	Number of people
AFR	air flow rate (l/s)
W	light rate (W)
PR	equipment input elec. Power
b	wind coefficient
E	percent fraction
Hp	horse power
L	length, crack
V	velocity
q	Heat loss or gain

### 3.3 GENERAL PROCEDURE

In estimating the heating load for the villa, the necessary steps can be outlined as follows:

- 1- Design value for outdoor winter is selected.
- 2- Indoor design conditions are selected on the activities to be carried out in the space.
- 3- The temperature in the unconditioned adjacent spaces is estimated. Usually the Average temperature between inside and outside is taken
- 4- Heat transfer coefficient and areas for the villa components are calculated from the villa plans and specifications. Surfaces between conditioned spaces must be omitted.

- 5- Rate of infiltration is estimated.
- 6- Rate of outside air required is estimated.
- 7- From the above data, transmission heat losses and infiltration and/or ventilation heat losses are computed these values are summed.
- 8- A certain pick-up load is required in order to reach the required condition in the space in a reasonable time.

### 3.4 PROCEDURE IN DETAILS

#### 3.4.1 Outdoor design conditions selection

These include dry-bulb temperature  $T_{out}$ , relative humidity  $\Phi_{out}$  and average air speed  $v$ . these values are usually tabulated local weather stations reports. Table 3-1 shows these values for Ram Allah town.

season	Dry-Bulb Temperature $T_{out}, ^\circ C$	Relative Humidity $\Phi, \%$	Average air speed m/s
Heating	0	50	3.0
Cooling	35	50	1.5

Table 3-1 shows values for out door design condition for Ram Allah town

#### 3.4.2 Indoor design conditions selection

Generally, an inside temperature equals  $22^\circ$  is taken as the inside design temperature in villa.

### 3.5 Heat transfer coefficient calculation

Usually heat transfer coefficients can be found for roof components, wall components, internal air, external air, and glass. Table 3-2 shows such values. Usually, the term "surface conductance",  $h(w/m^2, ^\circ c)$  is used for air and the term "Thermal resistance",  $R(m^2.c/w)$  is used with building materials, insulation and glass.



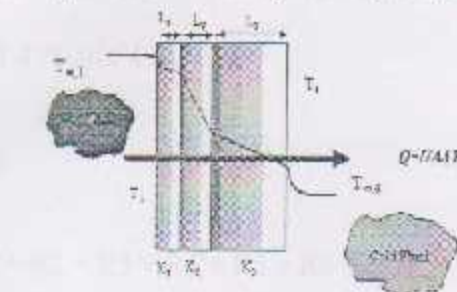
To find the overall heat transmission coefficients, this equation is used:

$$\frac{1}{U_{overall}} = R + 1/h \quad (3-1)$$

component	Members of component	Resistance (R) m <sup>2</sup> .c°/w	Surface conductance (h) w/m <sup>2</sup> .c°
roof	Plaster, 1.5cm		10.11
	Hollow brick, 20cm		5.11
	10cm		7.94
	Concrete, 8cm		9.0
	Polystyrene, 25mm		0.92
	Rubber		3.2
	Mortar, 2cm		36
	Tile, 2.5cm insulation		3.01
			1.36
External wall	Hollow brick, 20cm		5.11
Internal wall	Plaster, 2cm Hollow brick, 15cm	0.112	6.0
Internal air	Roof air		9.26
	Wall air		8.29
	Under floor air		6.13
External air	Any where		(winter)34.08 (summer)22.71
window	Single class		6.39

Table 3-2 shows values of Heat transmission coefficient Used in calculating heat losses or gain

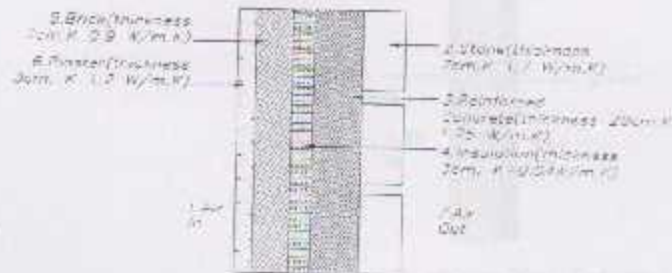
The overall heat-transfer coefficient U is a function of the thermal resistances; so we can calculate the values of thermal resistance for commonly used building materials enclosed air spaces and building envelope boundaries



$$Q \rightarrow \frac{1}{h} \quad \frac{l_1}{k_1} \quad \frac{l_2}{k_2} \quad \frac{l_3}{k_3} \quad \frac{1}{h}$$

Fig 3.1 Convection and conduction heat transfer

### 3.5.1 Calculation of external walls U value



Section For External Wall

$$R1 = 0.12 \text{ (m}^2\text{.K/W)}$$

$$R2 = d2 / k2 = 0.07 / 1.7 = 0.04 \text{ (m}^2\text{.K/W)}$$

$$R3 = d3 / k3 = 0.20 / 1.75 = 0.114 \text{ (m}^2\text{.K/W)}$$

$$R4 = d4 / k4 = 0.03 / 0.04 = 0.75 \text{ (m}^2\text{.K/W)}$$

$$R5 = d5 / k5 = 0.1 / 0.9 = 0.111 \text{ (m}^2\text{.K/W)}$$

$$R6 = d6 / k6 = 0.03 / 1.2 = 0.025 \text{ (m}^2\text{.K/W)}$$

$$R7 = 0.06 \text{ (m}^2\text{.K/W)}$$

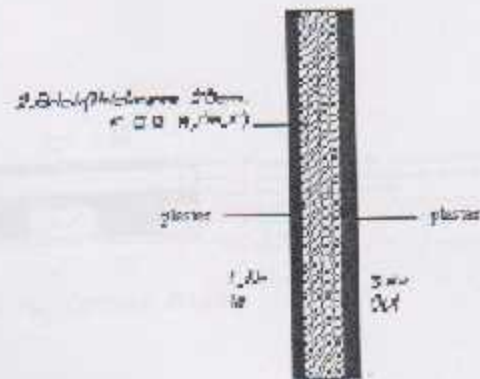
$$U_{\text{overall}} = 1 / \sum (R1 + R2 + R3 + R4 + R5 + R6 + R7)$$

$$= 1 / \sum (0.12 + 0.04 + 0.114 + 0.75 + 0.111 + 0.025 + 0.06)$$

$$= 1 / 1.22$$

$$= 0.82 \text{ (W/m}^2\text{.K)}$$

### 3.5.2 Calculation of internal walls U value



*Section For Internal Wall*

$$R_1 = 0.12 \text{ (m}^2\cdot\text{K/W)}$$

$$R_2 = d_2 / K_2 = 0.20 / 0.90 = 0.22 \text{ (m}^2\cdot\text{K/W)}$$

$$R_3 = 0.12 \text{ (m}^2\cdot\text{K/W)}$$

$$U_{\text{overall}} = 1 / \sum (R_1 + R_2 + R_3)$$

$$= 1 / \sum (0.12 + 0.22 + 0.12)$$

$$= 1 / 0.46$$

$$= 2.17 \text{ (W/m}^2\cdot\text{K)}$$

### 3.5.3 Calculation of exposed roofs U value



For section A:

$$R1 = 0.12 \text{ (m}^2\text{.K/W)}$$

$$R2 = d2 / k2 = 0.02 / 0.70 = 0.0286 \text{ (m}^2\text{.K/W)}$$

$$R3 = d3 / k3 = 0.05 / 1.75 = 0.286 \text{ (m}^2\text{.K/W)}$$

$$R4 = d4 / k4 = 0.20 / 1.75 = 0.114 \text{ (m}^2\text{.K/W)}$$

$$R5 = d5 / k5 = 0.02 / 1.2 = 0.017 \text{ (m}^2\text{.K/W)}$$

$$R6 = 0.06 \text{ (m}^2\text{.K/W)}$$

$$U_{\text{overall}} = 1 / \sum (R1 + R2 + R3 + R4 + R5 + R6)$$

$$= 1 / \sum (0.12 + 0.0286 + 0.286 + 0.114 + 0.017 + 0.06)$$

$$= 1 / 0.63$$

$$= 1.59 \text{ (W/m}^2\text{.K)}$$

For section B:

$$R1=0.12 \text{ (m}^2\text{.K/W)}$$

$$R2=d2 / k2 =0.02/0.70 =0.0286 \text{ (m}^2\text{.K/W)}$$

$$R3=d3 / k3 =0.05/1.75 =0.286 \text{ (m}^2\text{.K/W)}$$

$$R4=d4 / k4 =0.06/1.75 =0.034 \text{ (m}^2\text{.K/W)}$$

$$R5=d5 / k5 =0.14/0.039 =3.59 \text{ (m}^2\text{.K/W)}$$

$$R6=d6 / k6 =0.02/1.2 =0.017 \text{ (m}^2\text{.K/W)}$$

$$R7=0.06 \text{ (m}^2\text{.K/W)}$$

$$U_{\text{ overall}}=1/ \sum (R1+ R2 + R3 + R4 + R5 + R6 + R7)$$

$$=1/ \sum (0.12+0.0286+0.286+0.034+3.59+0.017+0.06)$$

$$=1/4.14$$

$$=0.24 \text{ (W/m}^2\text{.K)}$$

$$U_{\text{ overall}}=U_{\text{ overall for section A}} \times 0.2 + U_{\text{ overall for section B}} \times 0.8$$

$$= 1.59 \times 0.2 + 0.24 \times 0.8$$

$$= 0.51 \text{ (W/m}^2\text{.K)}$$

### 3.5.4 Calculation of ceiling U value



Section for ceiling

$$R1=0.12 \text{ (m}^2\text{.K/W)}$$

$$R2=d2 / K2 =0.02/1.05 =0.019 \text{ (m}^2\text{.K/W)}$$

$$R3=d3 / k3 =0.13/0.30 =0.43 \text{ (m}^2\text{.K/W)}$$

$$R4=d4 / k4 =0.02/0.04 =0.04 \text{ (m}^2\text{.K/W)}$$

$$R5=d5 / k5 =0.06/1.75 =0.164 \text{ (m}^2\text{.K/W)}$$

$$R6=d6 / k6 =0.14/0.95 =0.147 \text{ (m}^2\text{.K/W)}$$

$$R7=d7 / k7 =0.02/1.2 =0.017 \text{ (m}^2\text{.K/W)}$$

$$R8=0.12 \text{ (m}^2\text{.K/W)}$$

$$U_{\text{overall}}=1/\sum (R1+ R2 + R3 + R4 + R5 + R6+ R7 + R8)$$

$$=1/\sum (0.12+0.019+0.43+0.04+0.164+0.147+0.017+0.12)$$

$$=1/1.06$$

$$=0.94 \text{ (W/m}^2\text{.K)}$$

### 3.5.5 Calculation of glass U value

$$R_{\text{glass}} = 1/U_{\text{glass}} = 1/3.7 = 0.27 \text{ (m}^2\text{.K/W)}$$

$$R_{\text{air in}} = 0.12 \text{ (m}^2\text{.K/W)}$$

$$R_{\text{air out}} = 0.06 \text{ (m}^2\text{.K/W)}$$

$$U_{\text{glass}} = 1/\sum(R_{\text{glass}} + R_{\text{air in}} + R_{\text{air out}})$$

$$= 1/\sum(0.27 + 0.12 + 0.06)$$

$$= 1/0.45$$

$$= 2.22 \text{ (W/m}^2\text{.K)}$$

Material	Thickness (m)	Thermal Conductivity (W/m.K)	Thermal Resistance (m <sup>2</sup> .K/W)
Single Glazing	0.004	1.0	0.0004
Double Glazing	0.008	1.0	0.0016
Triple Glazing	0.012	1.0	0.0024
Low E Glass	0.004	1.0	0.0004

### 3.6 Ventilation Requirements

The ventilation requirements are listed in table 3-4 for different spaces. Table 3-4 shows ventilation standards for various applications.

Application	Class of Room
Meeting rooms	1.0
Offices	0.8
Classrooms	0.6
Stores	0.5
Class rooms	0.5
Work	0.5

Table 3-4 shows values of Ventilation standards for various applications.

### 3.6 Infiltration Rate Calculation

The air flow due to infiltration is calculated from the equation

$$Q = L[a\Delta T + b.v^2]^{\frac{1}{2}} \quad (3-2)$$

Where

Q= Air flow  $m^3/h$

L= crack area  $cm^2$ , it can be found in special tables as Shown in Table 3-3

a= stack coefficient  $(m^3/h)^2(cm)^{-4}(K)^{-1}$ , and taken as 0.00564 for Villa

b= wind coefficient  $(m^3/h)^2(cm)^{-4}(m/s)^{-2}$ , and taken as 0.00494 for light local shielding.

$\Delta T$ =temperature difference, K

v= average wind speed, m/s

Component	Infiltration leakage area ( $cm^2/cm^2$ )
External doors	11
Door frame	1.7
Windows	5.2
Window frame	1.7
Extract fan	20

Table 3-3 shows values of Leakage infiltration area  $cm^2/cm^2$

### 3.7 Ventilation requirements

The outdoor air requirements are listed in references for different spaces. Table 3-4 shows ventilation standard for sum applications.

Application	cfm/ft <sup>2</sup> of floor
Meeting rooms	1.0
Cafeteria	1.0
Kitchen	4.0
Store	1.0
Class room	0.5
toilet	2.0

Table 3-4 shows values of Ventilation standard for residences



### 3.8 Total calculation

Transmission loads can be calculated using this equation:

$$q = U_{overall} \cdot A \cdot \Delta T \quad (3-3)$$

Where:

$q$  = heat loss (w)

$U_{overall}$  = overall heat transmission coefficient ( $W/m^2 \cdot ^\circ C$ )

$A$  = area of heat transfer ( $m^2$ )

$\Delta T$  = temperature difference ( $^\circ C$ )

To find ventilation and infiltration loads, the following equation is used:

$$q = 1.2 \dot{V} \Delta T \quad (3-4)$$

Where

$\dot{V}$  is outdoor air flow in L/s

So, now we can calculate the heating load using this method for each room in the villa and we can choose the radiator in each room in each floor in the villa as in the following

For Basement floor

ROOM NAME	TOTAL LOAD IN Kcal
MULTIPURPOSE HALL	5701
GUEST ROOM	1398
BATH ROOM (1)	734
MAID ROOM	1071
BATH ROOM (2)	475
KITCHEN	1540
	<b>TOTAL LOAD = 10.919 Kcal=12.70 KW</b>

Table 3-5 loads in each room in the basement floor

RAD.NO.	NO. OF SECTION	RAD LENGTH CM	TYPE
1	11	100	AL 600
2	11	100	AL 600
3	11	100	AL 600
4	11	100	AL 600
5	11	100	AL 600
6	6	60	AL 600
7	9	84	AL 600
8	4	44	AI. 600
9	12	108	AI. 600
<b>TOTAL.</b>	<b>86 SEC.</b>	-----	-----

Table 3-6 numbers of sections for each radiator in the basement floor

**For Ground floor**

ROOM NAME	TOTAL LOAD IN Kcal
RECEPTION AND DINNING	6386
LOBBY (1)	622
WATER CIRCUIT	616
OFFICE	1626
BATH ROOM	490
ROOM BESIDE ELEVATOR	1549
BREAKFAST & LIVIVING ROOM	6429
LOBBY (2)	804
ENTARANCE LOBBY	887
<b>TOTAL LOAD = 19.409 Kcal = 22.57 KW</b>	

**Table 3-7 loads in each room in the ground floor**

RAD.NO.	NO. OF SECTION	RAD LENGTH CM	TYPE
1	18	156	AL 600
2	16	140	AL 600
3	16	140	AL 600
4	5	54	AL 600
5	5	54	AL 600
6	14	124	AL 600
7	4	44	AL 600
8	12	108	AL 600
9	18	156	AL 600
10	16	140	AL 600
11	16	140	AL 600
12	7	68	AL 600
13	7	68	AL 600
<b>TOTAL</b>	<b>154 SEC.</b>	-----	-----

**Table 3-8 numbers of sections for each radiator in the ground floor**

**For First floor**

ROOM NAME	TOTAL LOAD IN Kcal
BATH(1)	520
M. BATH	850
BATH (2)	650
MASTER BED ROOM	2370
WALK IN CLOSET(1)	619
LOBBY (1)	493
BED ROOM	1960
LOBBY (2)	2457
LIVING ROOM	5900
BED ROOM(1)	2151
BATH(3)	482
BED ROOM(2)	1720
WALK IN CLOSET(2)	850
<b>TOTAL LOAD = 21022 Kcal = 24.45 KW</b>	

**Table 3-9 loads in each room in the first floor**

RAD.NO.	NO. OF SECTION	RAD LENGTH CM	TYPE
1	4	44	AL 600
2	7	68	AL 600
3	5	52	AL 600
4	18	156	AL 600
5	5	52	AL 600
6	4	44	AL 600
7	15	84	AL 600
8	19	164	AL 600
9	14	124	AL 600
10	11	100	AL 600
11	11	100	AL 600

12	17	148	AL 600
13	4	44	AL 600
14	14	124	AL 600
15	7	68	AL 600
<b>TOTAL</b>	<b>155 SEC.</b>	<b>-----</b>	<b>-----</b>

**Table 3-10 numbers of sections for each radiator in the first floor**

So, now we can calculate the heating load using this method for each floor and the result as following:-

Floor NAME	TOTAL LOAD IN KCAL
BASMENT FLOOR	10.919
GROUND FLOOR	19.409
FIRST FLOOR	21.022
	<b>TOTAL LOAD=51.35 Kcal =59.71 KW</b>

**Table 3-11 total loads in each floor in the villa**

### 3.9 Pick-up load

For intermittently heated buildings, or in the case of the night thermostat setback, additional heat is required for raising the temperature of air, building materials and material contents of a building to the specified indoor temperature. For temperature difference of 17°C and less, for the system to reach its design temperature in 1 hour, 20% of design heat loss is enough.

### 3.10 Heating Boiler calculations & Selection

For boiler capacity = total no. of all sections in floors × heat transfer per section

$$=395 \text{ sec.} \times 130(\text{Kcal/h})$$

$$=51,350 \text{ (Kcal/h)}$$

We add 8600(Kcal/h) for hot water cylinder with 200 l.

So, using the following equation

$$Q= m \cdot c_p \cdot \Delta T$$

$$Q= 200 \cdot 1 \cdot 50 \cdot 0.86 = 8,600 \text{ Kcal/h}$$

Note: We use 11,400 (Kcal/h) as a safety factor

$$\text{Total boiler capacity} = 51,350 + 8,600 + 11,400$$

$$= 71,350 \text{ (Kcal/h)}$$

$$\text{Boiler capacity in (kW)} = (71,350 \times 1.2) / 860$$

$$= 99.56 \text{ (kW)}$$

$$\approx 100 \text{ (kW)}$$

### From catalogue

Using Boiler of the type **Fondital cast-iron boiler mod. ALOR 120**(See Appendix A) of the following specifications:-

Thermal Output=124.8 KW

Maximum thermal power =114.0KW



**Fig3.2 Boiler**

### 3.11 Radiator calculation & Selection

Using radiators of the type Fondital Die-cast aluminum radiators GB 600/100 with capacity of one section = 130 kcal (See Appendix A)

So, in order to calculate the numbers of sections of each radiator for any room:

No. of sections = heating load for the room (Kcal/h) / heat transfer per section

For example: for the master room in first floor, the heating load is 2.593(KW), heat transfer per section is 130(Kcal/h), so:

Heating load in (Kcal/h) =  $2.593 \times 860$

$$= 2230$$

No. of sections =  $2230(\text{Kcal/h}) / 130(\text{Kcal/h})$

$$= 17.2 \approx 17 \text{ sections}$$

Width of the radiator = (no of sections  $\times$  width for each section) + valves for both sides

Width for each section = 8cm

Valves for both sides = 12cm

Width of the radiator =  $(17 \times 8) + 12$

$$= 148\text{cm}$$

### 3.12 Collector capacity calculations & Selection

-There are 86 sections for basement floor

So

Collector capacity = no of sections  $\times$  heat transfer per section

$$= 86 \text{sec} \times 130 (\text{Kcal/h})$$

$$= 11,180 (\text{Kcal/h})$$

Collector capacity in kW =  $Q (\text{Kcal/h}) / 860$

$$= 11,180 / 860$$

$$= 13 \text{ kW}$$

#### 3.12.1 Flow rate & collector size calculations

Flow rate =  $(Q (\text{kW}) \times 0.24) / \Delta T$

$$= (13 \text{ kW} \times 0.24) / 15$$

$$= 0.208 \text{ L/s}$$

Flow rate =  $V \times A$

Assume  $V = 0.9 \text{ m/s}$

Flow rate =  $V \times ((\pi \times d^2) / 4)$

$$= 0.9 \times ((\pi \times d^2) / 4)$$



$$d = \sqrt{(4 \times \text{flow} / \pi \times 0.9)}$$

$$d = \sqrt{\text{flow} \times 1.19}$$

$$\text{Collector diameter} = \sqrt{\text{flow} \times 1.19}$$

$$= \sqrt{0.208 \times 1.19}$$

$$= 0.456 \times 1.19$$

$$= 0.54''$$

0.54'' approximated to 0.75'' diameter

- There are 154 sections for ground floor

So

Collector capacity = no of sections  $\times$  heat transfer per section

$$= 154 \text{ sec} \times 130 \text{ (Kcal/h)}$$

$$= 20,020 \text{ (Kcal/h)}$$

Collector capacity in kW =  $Q \text{ (Kcal/h)} / 860$

$$= 20,020 / 860$$

$$= 23.28 \text{ kW}$$

Flow =  $(Q \text{ (kW)} \times 0.24) / \Delta T$

$$= (23.28 \text{ kW} \times 0.24) / 15$$

$$Q = 0.37 \text{ L/s}$$

$$\text{Collector diameter} = \sqrt{\text{flow} \times 1.19}$$

$$= \sqrt{0.37 \times 1.19}$$

$$= 0.608 \times 1.19$$

$$= 0.72''$$

0.72'' approximated to 0.75'' diameter

3.13 Collector Capacity for multiple radiators

**-There are 155 sections for first floor**

So

$$\text{Collector capacity} = \text{no of sections} \times \text{heat transfer per section}$$

$$= 155 \text{ sections} \times 130 \text{ (Kcal/h)}$$

$$= 20,150 \text{ (Kcal/h)}$$

$$\text{Collector capacity in kW} = Q \text{ (Kcal/h)} / 860$$

$$= 20,150 / 860$$

$$= 23.43 \text{ kW}$$

$$\text{Flow} = (Q \text{ (kW)} \times 0.24) / \Delta T$$

$$= (23.43 \text{ kW} \times 0.24) / 15$$

$$= 0.37 \text{ L/s}$$

$$\text{Collector diameter} = \sqrt{\text{flow} \times 1.19}$$

$$= \sqrt{0.37 \times 1.19}$$

$$= 0.608 \times 1.19$$

$$= 0.72''$$

0.72'' approximated to 0.75'' diameter

### 3.13 Chimney Calculations & Selection

$$\text{Chimney capacity} = (\text{boiler (kW)} \times 12000) / 3.5$$

to obtain the capacity in Btu/h

$$= (100(\text{KW}) \times 12000) / 3.5$$

$$= 342857.14 \text{ Btu/h}$$

So, V type and head 50ft we choose chimney with 8'' diameter

### 3.14 Pump Calculation & Selection

Pump is selected based on:

- 1- Water flow rate (GPM)
- 2- Pump horse power (hp) – friction head

$$hp = \frac{\Delta p \cdot Q}{550} \quad (2-1)$$

Where,

$\Delta p$  = pressure loss lb/ft<sup>2</sup>

Q= water flow rate ft<sup>3</sup>/s

Eq.-(3-1) can be put as:

$$hp = \frac{\Delta p(lb/in^2) * 144 * GPM}{550 * 60} (0.1337)$$

Or

$$hp = \frac{\Delta p(lb/in^2) * GPM}{1714} \quad (3-5)$$

### 3.14.1 Pump calculations

First of all we calculate the flow rate of the entire floor, let's take the ground floor:

No of sections required in this floor is=129 section.

Heating capacity of this floor=no. of sections × 130 (Kcal/h)

$$=129 \times 130 \text{ (Kcal/h)}$$

$$=16,770 \text{ (Kcal/h)}$$

Heating capacity in KW= heating capacity (Kcal/h) / 860

$$=16,770 \text{ (Kcal/h)} / 860$$

$$=19.5 \text{ kW}$$

Flow (L/s) = (heating capacity (kW) × 0.24) / 15

$$= (19.5 \times 0.24) / 15$$

$$= 0.312 \text{ L/s}$$

$$\text{Flow (m}^3/\text{h)} = \text{Flow (L/s)} \times 3.6$$

$$= 0.312 \times 3.6$$

$$= 1.12 \text{ m}^3/\text{h}$$

$$\text{Head} = (\text{loop length (m)} \times 1.8 \times 450) / 12000$$

$$= (80 \times 1.8 \times 450) / 12000$$

$$= 5.4 \text{ m}$$

For safety:

$$\text{Head} = 5.4 \times 1.2$$

$$= 6.48 \text{ m} \approx 7 \text{ m}$$

Then by comparing this result with the pump curve we can select the suitable pump.

So, we select pump with flow rate = 1.12 m<sup>3</sup>/h and head = 7m

So, we use pump of the type **BIRAL PUMP NRB 15T** (medium-speed circulators with 3 speeds for heating and air conditioning) (Sec Appendix A)

### 3.15 Expansion tank calculation & selection

An expansion tank is used to maintain system pressure by allowing the water to expand when water temperature increases. It is normally required in a closed system. ASME has standardized the calculation of the capacity of closed expansion tanks. Water temperatures below 160°F use the following formula to determine the tank capacity:

$$V_t = \frac{E * V_s}{\frac{P_a - P_a}{P_f - P_o}} \quad (4-3)$$

Where:

$V_t$  = minimum capacity of the tank (gallons)

$E$  = percent increase in the volume of water in the system.

$V_s$  = total volume of water in the system (gallons)

$P_a$  = pressure in the expansion tank when water first enters, usually atmospheric pressure (ft of water absolute)

$P_f$  = initial fill or minimum pressure of expansion tank (ft of water absolute)

$P_o$  = maximum operating pressure at the expansion tank (ft of water absolute).

### 3.15.1 Expansion tank calculations

Minimum capacity of the tank (gallons) =

$$V_t = \frac{E * V_s}{\frac{P_a - P_a}{P_f - P_o}}$$

Where as:

$E$  at 70° = 2 %

By calculating the total flow of all the floors in l/s and convert to gallons per minute by multiply 15.852 we can get the total volume in gallons:

$V_s$  = 170 gallon

$P_a$  = 1 bar

$P_f$  = 2.2 bar

### 3.16 Example of Calculations:

$$P_0 = 5 \text{ bar}$$

$$V_t = (0.02 \cdot 170) / ((1/2.2) - (1/5))$$

$$= 3.4 / 0.25$$

$$= 13.6 \text{ gallons}$$

$$\text{Minimum capacity of the expansion in liter per second} = 13.6 \cdot 3.6$$

$$= 49 \text{ l/s}$$

We take it 80 l/s which are available.

*Handwritten mark*

### 3.16 Example of Calculation:-

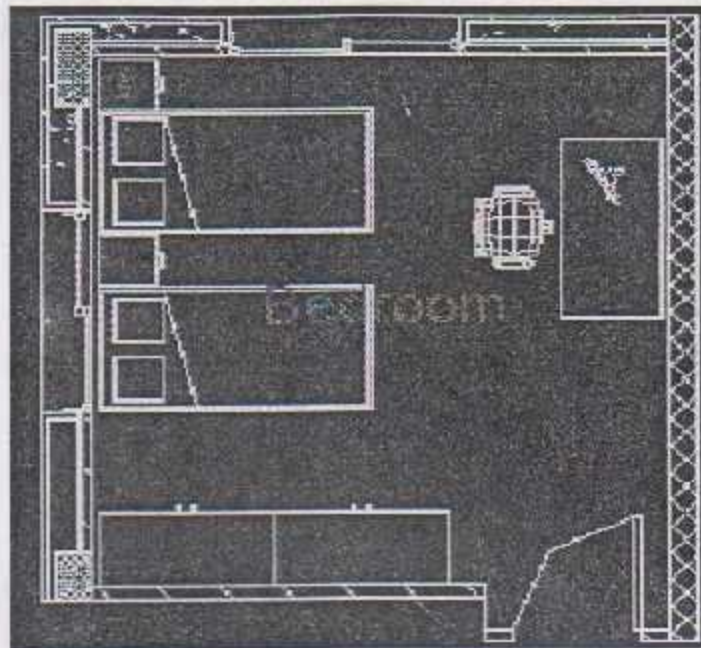


Fig3.3 Bed Room Plane

In order To calculate the load of the above Bed Room at first floor we must calculate the following:-

#### 1- Heat transfer through walls:-

##### A- For north and west walls (external walls):-

$$q = U_{overall} \cdot A \Delta T$$
$$= 0.82 \cdot 11.8(22-0) + 0.82 \cdot 12.8(22-0) = 213 + 231 = 444 \text{ W}$$

##### B- For south wall (internal wall):-

We consider that there is no heat transfer because there is no temperature difference between heating spaces (all adjacent spaces have the same temperature as this room)



### C- For cast wall (interior walls):-

We consider the temperature of the stair case is the average temperature between inside design temperature ( $T_i$ ) and out side design temperature ( $T_o$ )

So,

$$\text{Stair case temperature} = (T_i + T_o)/2$$

$$= (22+0) / 2 = 11^\circ\text{C}$$

Then

$$q = U_{\text{overall}} \cdot A \cdot \Delta T$$

$$= 2.17 \cdot 14.4 \cdot (22-11) = 344 \text{ W}$$

### 2- Heat transfer through Roof:-

$$q = U_{\text{overall}} \cdot A \cdot \Delta T$$

$$= 0.51 \cdot 17.3 \cdot (22-0) = 194 \text{ W}$$

### 3-Heat transfer through Floor:-

We consider that there is no heat transfer because there is no temperature difference between heating spaces (ground floor have the same temperature as this first floor)

### 4-For north and west glasses:-

$$q = U_{\text{overall}} \cdot A \cdot \Delta T$$

$$= 2.22 \cdot 1.6 \cdot (22-0) + 2.22 \cdot 1.6 \cdot (22-0) = 156 \text{ W}$$

### 5- Heat transfer through infiltration :-

We will calculate heat load through infiltration by knowing the dimension of the windows that cause infiltration

$$Q_{inf} = 862 \text{ W}$$

Then the total heating load equal the load through walls, Roof, floor, glasses, doors and loads through infiltration

So,

$$Q_{total} = Q_{Trans} + Q_{inf}$$

$$Q_{Trans} = Q_{transmission} = Q_{walls} + Q_{Roof} + Q_{floor} + Q_{wind} + Q_{door}$$

$$Q_{Trans} = 444 + 194 + 156 + 344 = 1138 \text{ W}$$

SO,

$$Q_{total} = Q_{Trans} + Q_{inf}$$

$$Q_{total} = 1138 + 862 = 2000 \text{ W}$$

So, in order to calculate the numbers of sections of each radiator for any room:

No. of sections = heating load for the room (Kcal/h) / heat transfer per section

So, for the bed room in first floor, the heating load is 2(kW), heat transfer per section is 130(Kcal/h), so:

$$\text{Heating load in (Kcal/h)} = 2 \times 860$$

$$= 1.72 \text{ (Kcal/h)}$$

$$\text{No. of sections} = 1.72 \text{ (Kcal/h)} / 130 \text{ (Kcal/h)}$$

$$=13.7 \approx 14 \text{ sec}$$

Width of the radiator = (no of sections  $\times$  width for each section) + valves for both sides

Width for each section = 8cm

Valves for both sides = 12cm

Width of the radiator =  $(14 \times 8) + 12$

$$= 124 \text{ cm}$$

## 4.1 INTRODUCTION

Plumbing consists of the pipes, which in water supply systems and drainage distribution system.

It is one of the most essential and vital services that is performed. This is every day and need to be aware.

# CHAPTER FOUR

## SELECTION OF PLUMBING SYSTEM

Plumbing is the art of installing and maintaining the water supply and drainage systems. The selection of a plumbing system depends on the nature of the service to be provided, the location of the building, the climate, the cost of the materials, and the availability of skilled labor.

The selection of a plumbing system is a complex task that requires a thorough understanding of the building's requirements and the local plumbing codes. The selection of a plumbing system is a complex task that requires a thorough understanding of the building's requirements and the local plumbing codes. The selection of a plumbing system is a complex task that requires a thorough understanding of the building's requirements and the local plumbing codes. The selection of a plumbing system is a complex task that requires a thorough understanding of the building's requirements and the local plumbing codes.



Fig. 4.1 Plumbing System

#### 4.1 INTRODUCTION:-

Plumbing consist of two things which is water supply system and drainage distribution system

It is one of the most essential and vital subject that its problems face us every day and need a fast solution to treat it

Most dwellings and buildings, including those in rural areas are supplied with water from a public water supply, otherwise known as the mains supply. The design of a mains water supply needs to consider present demand and anticipated future demand, the size of the water mains, and the pressure of water in the mains which is known as the 'head', the height to which the water would rise in a vertical pipe.

The connection to the mains water supply is usually taken to the boundary of the site and finished with a stop valve or stop cock, housed in a suitable box or purpose chamber. This chamber may be fitted with a hinged cast iron cover. The cold water supply for the dwelling is taken from the stop valve to the building, 750 mm below ground level. A second stop valve should be fitted on the service pipe where it enters the building. Where possible this should be at the kitchen sink, although the location is not critical. Inside the house a drain cock should be fitted above the stop valve to allow the cold water system to be drained down.

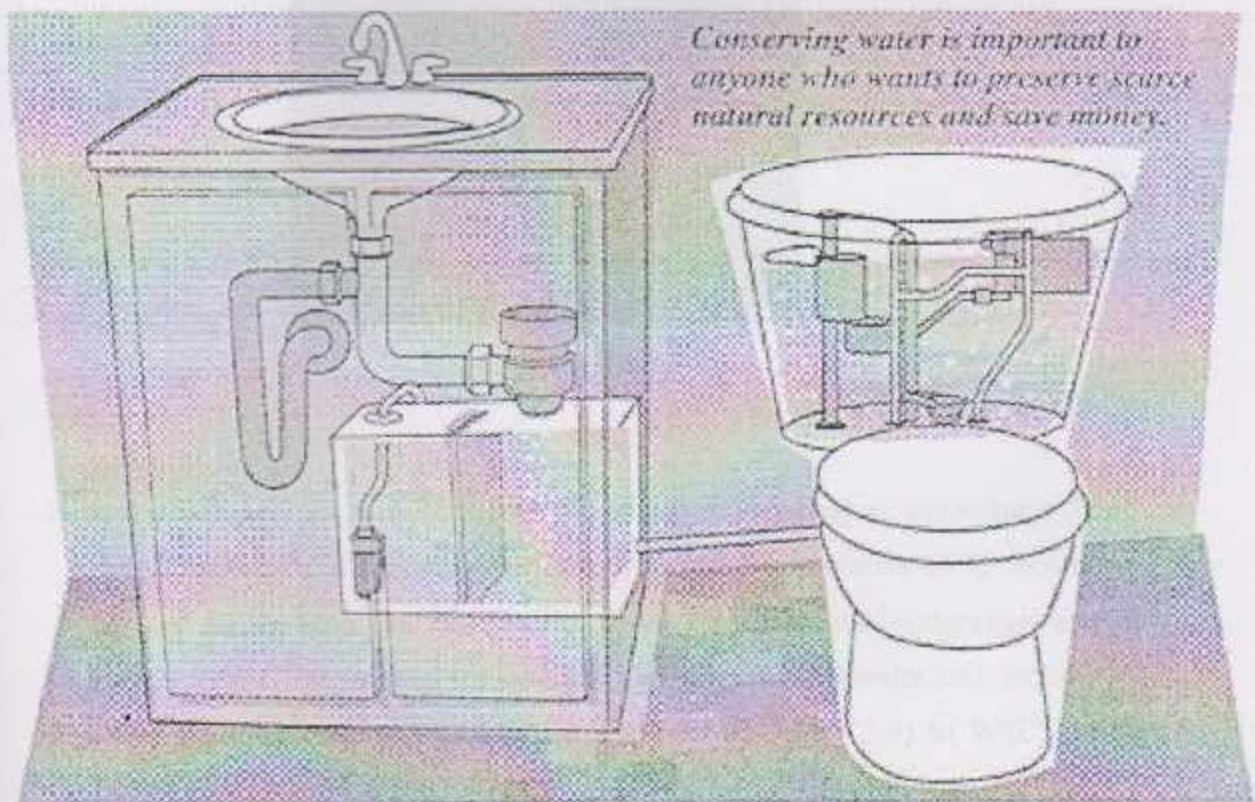


Fig 4.1 Plumbing System

## 4.2 Water System

### 4.2.1 Calculations For hot and cold water:-

To determine the pipe size for cold and hot water we must calculate the water supply fixture unit (WSFU) for each fixture and fixture unit total on each piping run out and determine the minimum flow pressure required at the most remote outlet.

We can determine the required pipe in each section using the friction head loss data calculated and the friction head chart as follows:-

**Example:** - to calculate the water supply fixture unit (WSFU) for each fixture in the Bath room shown

We have three fixtures (lavatory, shower, water closet with flush tank) each have (WSFU) as follow:-

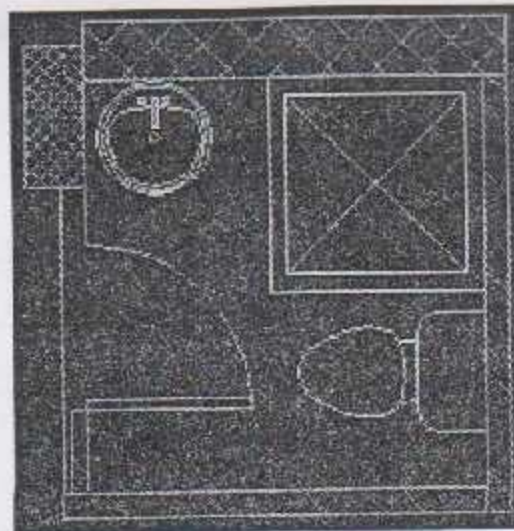


Fig 4.2 Bath Room

- 1- lavatory is a fixture with both cold and hot water supplies, so, the weights for maximum separate demands may be taken as three-quarters (3/4) the list demand for the supply in Table (4.1),so from Table (4.1) the lavatory (private) gives 1 WSFU for the total demand (for both cold and hot water ) so, for cold water only or hot water only we take  $(3/4 * WSFU = 3/4 * 1 = 3/4)$  & ( WSFU =1 for both cold and hot water)

2- shower is a fixture with both cold and hot water supplies, so, the weights for maximum separate demands may be taken as three-quarters (3/4) the list demand for the supply in Table (4.1), so from Table (4.1) the shower (private) gives 2 WSFU for the total demand (for both cold and hot water ) so, for cold water only or hot water only we take  $(3/4 * WSFU = 3/4 * 2 = 1.5)$  & ( WSFU =2 for both cold and hot water)

3- Water closet is a fixture with cold water supply only, so, the weights for maximum separate demands may be taken as three-quarters (3/4) the list demand for the supply in Table (4.1),so from Table(4.1)the water closet (private and with flush tank) gives 3 WSFU for the total demand (for both cold and hot water ) so, for cold water only or hot water only we take  $(3/4 * WSFU = 3/4 * 3 = 2.25)$  & ( WSFU =3 for both cold and hot water)

4- So, from the above information we can do the following table:-

Fixture Unit	No. of Units	WSFU From table(9..3)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot & Cold Water
Lavatory (private)	1	$3/4 * 1$	3/4	3/4	1
Shower (private)	1	$3/4 * 2$	1.5	1.5	2
Water Closet flush tank(private)	1	$3/4 * 3$	2.25	-----	2.25
-----	-----	-----	$\Sigma = 4.5$ WSFU	$\Sigma = 2.25$ WSFU	$\Sigma = 5.25$ WSFU

Table 4-1 WSFU for the Bath Room

Now we can calculate the number of fixture in each floor from Mechanical Drawing and so we can do the following table:-

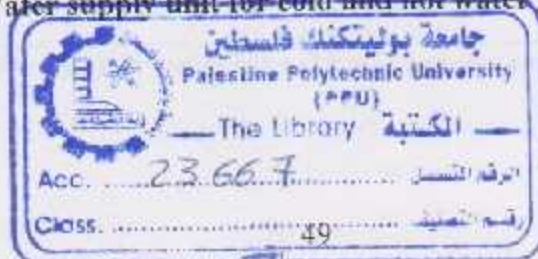
Floor name	Total Lavatory	Total sink	Total Bath tube	Total Shower head	Total Water closet /flush tank
Basement	3	1	1	2	3
Ground floor	2	1	1	-----	2
First floor	4	1	3	1	3
Continuous flow at garden	3/4"	-----	-----	-----	-----

Table 4-2 Total number for water supply fixture unit for cold and hot water

Now we can calculate the (WSFU) for each floor, So (WSFU) for each floor as in the following tables:

Fixture Unit	No. of Units	WSFU From table (4.1)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot Water Cold Water &
<b>BASMENT FLOOR</b>	-----	-----	-----	-----	-----
Lavatory (private)	3	$\frac{3}{4} * 1$	2.25	2.25	3
Kitchen Sink(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Bath tube(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Water Closet flush tank(private)	3	$\frac{3}{4} * 3$	6.75	-----	6.75
Continuous flow for Garden	1	$\frac{3}{4} * 3$	2.25	-----	3
-----	-----	-----	$\Sigma = 14.25$ WSFU	$\Sigma = 5.25$ WSFU	$\Sigma = 15.75$ WSFU

Table 4-3 Water supply unit for cold and hot water for basement floor





Fixture Unit	No. of Units	WSFU From table (4.1)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot Water Cold Water & Water &
<b>GROUND FLOOR</b>	-----	-----	-----	-----	-----
Lavatory (private)	2	$\frac{3}{4} * 1$	1.5	1.5	2
Kitchen Sink(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Bath tube(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Water Closet flush tank(private)	2	$\frac{3}{4} * 3$	4.5	-----	4.5
-----	-----	-----	$\Sigma = 9$ WSFU	$\Sigma = 4.5$ WSFU	$\Sigma = 10.5$ WSFU

Table 4-4 Water supply unit for cold and hot water for ground floor

Fixture Unit	No. of Units	WSFU From table (9.3)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot Water Cold Water & Water &
<b>FIRST FLOOR</b>	-----	-----	-----	-----	-----
Lavatory (private)	4	$\frac{3}{4} * 1$	3	3	4
Kitchen Sink(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Bath tube(private)	3	$\frac{3}{4} * 2$	4.5	4.5	6
Water Closet flush tank(private)	3	$\frac{3}{4} * 3$	6.75	-----	6.75
-----	-----	-----	$\Sigma = 15.75$ WSFU	$\Sigma = 9$ WSFU	$\Sigma = 19.25$ WSFU

Table 4-5 Water supply units for cold and hot water for first floor

#### 4.2.2 Flow rate calculations

To calculate flow rate in gpm:-

##### FOR BASMENT FLOOR

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank for basement floor the estimating demand in gpm for cold water = 14.25 WSFU.

So by using interpolation = 10.55 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for basement floor estimating demand in gpm for hot water = 5.25 WSFU.

So by using interpolation = 4.25 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for basement floor estimating demand in gpm for hot & cold water = 15.75 WSFU.

So by using interpolation = 11.55gpm

**Note:** - For supply outlets likely to impose continuous demand, we estimate continuous separately and add to the total demand for fixture

We have 1 faucets (Ø 3/4) for garden so from Table 4-2(See Appendix B)

The total flow in gpm = 2.25 WSFU. = 2 gpm

So the total cold and hot water demand needed = 11.55 + 2 = 13.55 gpm

## FOR GROUND FLOOR

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank for ground floor the estimating demand in gpm for cold water = 9 WSFU.

So by using interpolation = 7.25 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for ground floor estimating demand in gpm for hot water = 4.5 WSFU.

So by using interpolation = 3.875 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for ground floor estimating demand in gpm for hot & cold water = 10.5 WSFU.

So by using interpolation = 8.4 gpm

So the total cold and hot water demand needed = 10.5 WSFU. = 8.4 gpm

## FOR FIRST FLOOR

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank for first floor the estimating demand in gpm for cold water = 15.75 WSFU.

So by using interpolation = 11.45 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for first floor estimating demand in gpm for hot water = 9 WSFU.

So by using interpolation = 7.25 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for first floor estimating demand in gpm for hot & cold water = 19.25 WSFU.

So by using interpolation = 13.48 gpm

So the total cold and hot water demand needed = 13.48 gpm

**To calculate static head for Basement floor for cold water:-**

Floor to floor height = 4.00

We have 3 floor (basement + ground+ first) and so  $4.0 \times 3.0 = 12.0$

Sink outlet above basement level = 1.05 m

& tank outlet above roof level = 0.75

So there is static head in this case =  $12 + 0.75 - 1.05 = 11.40 \text{ m} = 16.22 \text{ psi}$

**To calculate the equivalent length:-**

Pumps at Basement floor transfer the water through pipes to the tank at Roof from which it distribute the water through pipes to the collectors at the First, Ground & Basement floor

So we will calculate the equivalent length from the tank at roof to the farthest outlet at the Basement floor (sink) at farthest collector

#### **FOR COLD WATER**

Total length through shaft room from Roof to the Basement floor = 16.0 m

Total length in the Basement floor from riser at the shaft room to farthest out let (sink) at farthest collector = 18.0 m

So, the total length from roof tank to the farthest outlet at Basement the floor (sink) at farthest collector

$$\text{Total length} = 18.0 + 16.0 = 34.0 \text{ m}$$

$$\text{Total length} = 18.0 + 16.0 = 34.0 \text{ m}$$

To calculate the equivalent length we assumed 50% additional equivalent length

to account for fittings there for the total developed length or the equivalent length as follows:

$$\text{Equivalent length} = \text{Total length} * 1.5$$

$$\text{Equivalent length} = 34 * 1.5 = 51 \text{ m}$$

$$\text{Equivalent length} = 51 * 3.28 = 167.28 \text{ ft}$$

**To calculate static head for Basement floor for hot water:-**

Sink outlet above basement level = 1.05 m

& heat exchanger outlet above basement level = 1.05 m

So there is no static head in this case and so static head equal zero

So there is no static head in this case and so static head equal zero

So there is no static head in this case and so static head equal zero

## FOR HOT WATER

Total length through shaft room from heat exchanger to the Basement floor = 16.0 m

Total length in the Basement floor from collector to farthest outlet (sink) at farthest point = 13.5 m

Total length = 16.0 + 13.5 = 29.5 m

So the total length from heat exchanger to the farthest outlet at Basement the floor (Sink) at farthest collector

$$\text{Total length} = 13.50 + 16.0 = 29.50 \text{ m}$$

To calculate the equivalent length we assumed 50% additional equivalent length

to account for fittings there for the total developed length or the equivalent length as follows:

Equivalent length = Total length \* 1.5

Equivalent length = 29.5 \* 1.5 = 44.25m

Equivalent length = 44.25 \* 3.28 = 145.14 ft

**The minimum required flow pressure:-**

The minimum required flow pressure at the most remote outlet on the Basement floor (sink) is 8.0 psi from table 4-2 (See Appendix B).

**To calculate the friction head**

For cold water

After calculating the above requirement we can calculate the friction head from the following equation:

Static Head = minimum required flow pressure + friction head

So friction head = 16.22 + 8.0 = 24.22 psi

For hot water

Static Head = minimum required flow pressure + friction head

So friction head = 0.0 + 8.0 = 8.0 psi

The usual design aims for uniform friction loss along the entire pipe length to do this we establish a friction loss per 100 ft by dividing total loss by total length and then size the pipe accordingly.

Uniform design friction loss in psi/100ft is:-

### FOR COLD WATER

Friction /100 ft = available friction head/ total equivalent length

$$\text{Friction /100 ft} = 24.22 \text{ psi} / 167.28 * 100 \text{ ft} = 14.48 \text{ psi} / 100 \text{ ft}$$

So for flow rate 14.80 gpm and friction head loss 14.48 psi / 100ft from figure 4-9 (See Appendix B) the diameter of the pipe from the Roof tank to the collector at Basement floor 1" and the velocity is 5.5 fps and the friction head is 12.0 psi / 100ft.

### FOR HOT WATER

Friction /100 ft = available friction head/ total equivalent length

$$\text{Friction /100 ft} = 8.0 \text{ psi} / 145.14 * 100 \text{ ft} = 5.51 \text{ psi} / 100 \text{ ft}$$

So for flow rate 4.25 gpm and friction head loss 5.51 psi / 100ft from figure 4-9 (See Appendix B) the diameter of the pipe from the heat exchanger to the collector at Basement floor 3/4" and the velocity is 2.5 fps and the friction head is 5.0psi / 100ft

Now we can determine pipe size, velocity of water and friction head for each floor as shown:-

	Equivalent length in (ft)	Pipe size in (inch)	Friction (psi/100ft)	Velocity(fps)	Section friction(psi)
nk to ior	167.28	1"	14.48	5.5	12.0
nk to r	113.16	1"	17.1	3.5	6.0
nk to	186.96	1.25"	8.56	3.2	3.5

Table 4-6 Pipe size, velocity of cold water and friction head for each floor

## 4.3 Sanitary Drainage system

### 4.3.1 Manhole Design

We design the manholes around the building so as that the sewage comes from the stacks flows in , then the sewage flows from one manhole to another so as reaching the septic tank.

The design of the manholes depend on the ground and its nature around the building, and so as the first manhole height should not be less than 50 cm. and then we calculate the height of the other manholes depending on the spacing between manholes and the slope of drainage pipes between manhole to be 1.2 % .

As a result of these calculations we estimate the invert level of the manhole that is the depth of the pipe entering the manhole. And we chose the diameter of the manhole depending on the depth of the manhole. As below

Φ60 cm for manhole depth (50-100) cm.

Φ80 cm for manhole depth (100-150) cm.

Φ100 cm for manhole depth (150-250) cm.

Φ120 cm for manhole depth > 250 cm

### 4.3.2 Manhole calculations

We assume the depth of the first manhole to be 50cm & we calculate the second manhole according to it and so on,

For manhole #2:

The distance between manhole #1 & manhole #2 is 9.8m,

Invert level for the manhole#2 is:

$$y = ((s \times \text{slope}) + 5)/100$$



Where:  $s$  is the distance between manhole#1 & manhole#2

Slope is 1.2

5cm is the point in manhole #2 where the pipe will be connected.

So:

$$y = (9.8 \times 1.2) + 5 / 100$$

$$= 0.168$$

We added to the invert level of manhole #1 to get the invert level to manhole#2

$$\text{Invert level for manhole#2} = -0.168 - 0.50$$

$$= -0.668 \approx -0.67$$

Then we calculate the depth of manhole#2:

$$\text{Depth} = \text{top level} - \text{invert level}$$

$$= 0.00 - (-0.67)$$

$$= 67\text{cm}$$

depth  $\rightarrow$  100cm,  $\Phi 60$

100  $\rightarrow$  150cm,  $\Phi 80$

150  $\rightarrow$  200cm,  $\Phi 100$

200  $\rightarrow$  ,  $\Phi 120-150$

So diameter of manhole#2 is 60

Manholes no.	Top level cm	Invert level cm	Depth cm	Diameter cm	Cover type
M1	0.0	- 0.5	50	60	Medium duty
M2	0.0	- 0.67	67	60	Medium duty
M3	0.0	-3.62	3.62	120	Medium duty
M4	- 3.42	-3.92	50	60	Medium duty
M5	-3.42	-4.00	62	60	Medium duty
M6	-3.38	-4.11	73	60	Medium duty
M7	-3.38	-4.20	82	60	Medium duty
M8	-3.38	-4.30	92	60	Medium duty
M9	-3.38	-4.32	94	60	Medium duty
M10	-2.97	-3.47	50	60	Medium duty
M11	-3.12	-4.52	140	80	Medium duty

Table 4-7 shows manholes levels

#### Example:-

Here we will talk about the choose of diameter and the slope of the drainage pipe system and we will take this Bath Room as an example of how we will choose the diameter and the slope of the drainage pipe system

1- We will use pipes (Branches) from fixture unit to the floor trap (F.T) with diameter (1.5-2)" for lavatory and shower and with slope (2%)

2- We will use pipes (Building drains) from fixture unit to the manhole with diameter (4)" for water closet with flush valve and with slope (1%-1.5%)

3- We will use pipes (Sewage pipes) between manholes with diameter (4)" and with slope (1.2%) and the waste water will transfer between manholes until it reach the main manhole.

4- We will use floor trap (F.T) at the end of the branches as a collection box for this pipes and in order to provide a water seal to prevent odors, sewage gases and vermin's from entering building

5- We will use clean out (C.O) at the end of the branches in order to clean the pipes from any things that can blockage and close the pipes

6- We will use a stack with diameter (4)" in order to drain the waste water to the manholes

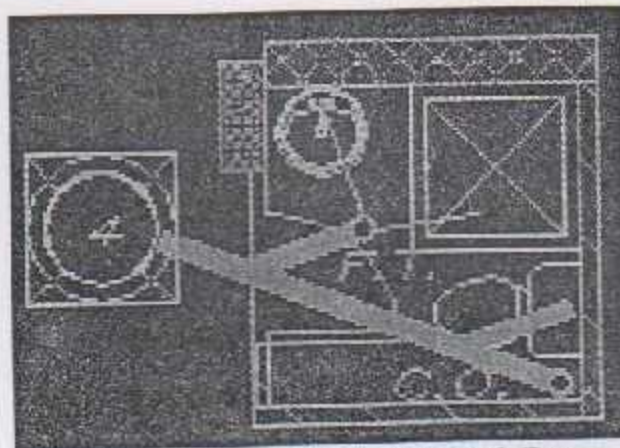


Fig 4.3 Sanitary Drainage for Bath Room

#### 4.3.3 Drainage piping fill

- 1) Branches are designed to run maximum of 50% fill.
- 2) Stacks are designed to flow between 25-33% maximum.
- 3) Building drains and sewer drains may be designed over 50% fill

#### 4.3.4 Drainage piping velocity

- 1) For branches the recommended velocity is 2 ft/s.
- 2) For building the recommended velocity is 3 ft/s.
- 3) For greasy the recommended velocity is 4 ft/s.

Velocity of water flow through drainage piping depends on

- 1) Pipe diameter
- 2) Slope

For the same diameter large pipe diameter required lower slope

For pipes of diameter  $\leq 3''$  the minimum slope is  $1/4$  in/ft

For pipes of diameter  $\geq 4''$  the minimum slope is  $1/8$  in/ft

## CONCLUSION And RECOMMENDATIONS

Accordingly, the following are suggested:

1. The existing syllabus should be revised to take into account the latest developments in the field of science. It should be more comprehensive and up-to-date. It should also be more relevant to the needs of the Indian society which is a rapidly changing one.

## CHAPTER FIVE

### CONCLUSION

#### And

### RECOMMENDATIONS

1. The present syllabus is comprehensive and up-to-date. It should be revised to take into account the latest developments in the field of science. It should also be more relevant to the needs of the Indian society which is a rapidly changing one.

## 8.1 Recommendations of Central heating system

According our study we notice that:

No	Description	Manufacturer/Company	Price per meter
1-	Heating load calculated from the calculation is less than the load calculated from our expert in market. So, the first one is more economic and it is sufficient. So, we will be install the no. of section of radiators which is a result of our calculation		
2-	The pipes diameter calculated is less than the pipes diameter chosen from our Expert so, we will use the pipes diameter which is a result from calculation		
3-	The selected boiler from the load calculated from the calculation is less in its capacity than the boiler calculated from our expert which means that the first one is more economic and sufficient. So, we will be install the Boiler with capacity which is a result of our calculation		

Table 5-1: Comparison of diameters

According our study we notice that:

- 1- Pipe that we calculated from our study is more economic than the pipe diameter chosen from our expert, and since that the first one is less than the second one which is more economic will it be selected

## 8.2 Recommendations of sanitary drainage system

- 1- The applied diameter for 4" diameter pipes the available pipes will be 4" diameter which is 4" diameter which is 4" diameter which is 4" diameter

- 2- For what concerning economical aspect we find that there is a big price difference in the price of pipe and fittings as long as we are talking about diameter 4"

## 8.2 Recommendations of hot and cold water system

No.	Description	Result of calculations	From our expert
1	DIAMETER OF PIPE OF COLD WATER FOR BASEMENT FLOOR	1"	1"
2	DIAMETER OF PIPE OF COLD WATER FOR GROUND FLOOR	1"	1"
3	DIAMETER OF PIPE OF COLD WATER FOR FIRST FLOOR	1.25"	1"
4	DIAMETER OF PIPE OF HOT WATER FOR FIRST FLOOR	3/4"	1"

Table 5-1 shows values of diameters

According our study and from the above table we notice that:

- 1- Pipe diameter calculated some time is nearly the same as the pipe diameter chosen from our experts, and some time the first one is less than the second one which means that it is more economic and it is sufficient

## 8.3 Recommendations of sanitary drainage system

- 1- Most applied diameters are 4" this due to that the available pipes and fitting in the total market are 4" diameter which safe
- 2- For what concerning economical aspect we find that there is no much difference in the price of pipe and fitting as long as we are taking a bout diameter  $\leq 4$ .

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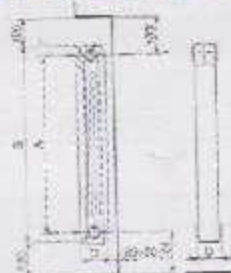




# APPENDIX A

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82	...	...
83	...	...
84	...	...
85	...	...
86	...	...
87	...	...
88	...	...
89	...	...
90	...	...
91	...	...
92	...	...
93	...	...
94	...	...
95	...	...
96	...	...
97	...	...
98	...	...
99	...	...
100	...	...

### 3.1 catalogue of radiator



# 100



## Dati tecnici

Modello	Altezza (mm)	Profondità (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)
350/100	37	427	350	40	0	0,50	1,12	82,31	1,2074	0,8220
400/100	37	575	400	40	0	0,50	1,43	130,85	1,5100	0,9174
450/100	37	675	450	40	0	0,50	1,75	149,33	1,5514	1,0368
500/100	37	775	500	40	0	0,50	2,07	165,21	1,5719	1,1574
600/100	37	977	600	40	0	0,50	2,74	183,52	1,5620	0,9590

Pressione massima di esercizio: 600 kPa (6 bar)  
 Esecuzione standard per modello: 3 e 4, 10°  
 I valori di potenza (kW) sono indicati in base a  $\Delta T = 50^\circ\text{C}$  e  $\Delta T_{\text{amb}} = 10^\circ\text{C}$



### 350/100

N° sezioni	Altezza (mm)	Profondità (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)
1	37	427	350	40	0	0,50	1,12	82,31	1,2074
2	37	427	350	40	0	0,50	1,12	82,31	1,2074
3	37	427	350	40	0	0,50	1,12	82,31	1,2074
4	37	427	350	40	0	0,50	1,12	82,31	1,2074
5	37	427	350	40	0	0,50	1,12	82,31	1,2074
6	37	427	350	40	0	0,50	1,12	82,31	1,2074
7	37	427	350	40	0	0,50	1,12	82,31	1,2074
8	37	427	350	40	0	0,50	1,12	82,31	1,2074
9	37	427	350	40	0	0,50	1,12	82,31	1,2074
10	37	427	350	40	0	0,50	1,12	82,31	1,2074

### 500/100

N° sezioni	Altezza (mm)	Profondità (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)
1	37	775	500	40	0	0,50	2,07	165,21	1,5719
2	37	775	500	40	0	0,50	2,07	165,21	1,5719
3	37	775	500	40	0	0,50	2,07	165,21	1,5719
4	37	775	500	40	0	0,50	2,07	165,21	1,5719
5	37	775	500	40	0	0,50	2,07	165,21	1,5719
6	37	775	500	40	0	0,50	2,07	165,21	1,5719
7	37	775	500	40	0	0,50	2,07	165,21	1,5719
8	37	775	500	40	0	0,50	2,07	165,21	1,5719
9	37	775	500	40	0	0,50	2,07	165,21	1,5719
10	37	775	500	40	0	0,50	2,07	165,21	1,5719

### 600/100

N° sezioni	Altezza (mm)	Profondità (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)
1	37	977	600	40	0	0,50	2,74	183,52	1,5620
2	37	977	600	40	0	0,50	2,74	183,52	1,5620
3	37	977	600	40	0	0,50	2,74	183,52	1,5620
4	37	977	600	40	0	0,50	2,74	183,52	1,5620
5	37	977	600	40	0	0,50	2,74	183,52	1,5620
6	37	977	600	40	0	0,50	2,74	183,52	1,5620
7	37	977	600	40	0	0,50	2,74	183,52	1,5620
8	37	977	600	40	0	0,50	2,74	183,52	1,5620
9	37	977	600	40	0	0,50	2,74	183,52	1,5620
10	37	977	600	40	0	0,50	2,74	183,52	1,5620

### 700/100

N° sezioni	Altezza (mm)	Profondità (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)
1	37	1179	700	40	0	0,50	3,41	201,83	1,5425
2	37	1179	700	40	0	0,50	3,41	201,83	1,5425
3	37	1179	700	40	0	0,50	3,41	201,83	1,5425
4	37	1179	700	40	0	0,50	3,41	201,83	1,5425
5	37	1179	700	40	0	0,50	3,41	201,83	1,5425
6	37	1179	700	40	0	0,50	3,41	201,83	1,5425
7	37	1179	700	40	0	0,50	3,41	201,83	1,5425
8	37	1179	700	40	0	0,50	3,41	201,83	1,5425
9	37	1179	700	40	0	0,50	3,41	201,83	1,5425
10	37	1179	700	40	0	0,50	3,41	201,83	1,5425

### 800/100

N° sezioni	Altezza (mm)	Profondità (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)	Spessore (mm)
1	37	1381	800	40	0	0,50	4,08	220,14	1,5230
2	37	1381	800	40	0	0,50	4,08	220,14	1,5230
3	37	1381	800	40	0	0,50	4,08	220,14	1,5230
4	37	1381	800	40	0	0,50	4,08	220,14	1,5230
5	37	1381	800	40	0	0,50	4,08	220,14	1,5230
6	37	1381	800	40	0	0,50	4,08	220,14	1,5230
7	37	1381	800	40	0	0,50	4,08	220,14	1,5230
8	37	1381	800	40	0	0,50	4,08	220,14	1,5230
9	37	1381	800	40	0	0,50	4,08	220,14	1,5230
10	37	1381	800	40	0	0,50	4,08	220,14	1,5230

### 3.2 catalogue of boilers

## CASTIRON BOILERS Mod. ALOR

### MAIN FEATURES

MODEL		80	100	120	140	160
Thermal output	W	27	137,4	124,8	144,3	169,8
Maximum thermal power	W	32	38	114	32	150
Minimum thermal power	W	27	93	83	107	110
Efficiency at nominal load	%	91	91,2	91,4	91,8	91,7
Efficiency at reduced load (30%)	%	92,2	92,9	92,8	93,5	92,7
EC efficiency class		★★	★★	★★	★★	★★
Load loss at $\Delta t = 50 K$	%	0,52	0,54	0,51	0,49	0,49
Flue gas temperature	°C	190	178	177	176	175
Flue gas temperature at minimum power	°C	183	164	157	164	161
Flue gas flow rate at maximum thermal capacity (lit)	kg/h	141	195	192	220	191
Flue gas flow rate at maximum thermal capacity (natural gas)	kg/h	145	179	201	236	200
Volume on the flue gas side	dm <sup>3</sup>	35,8	108,2	109	143,9	102,6
Combustion chamber volume	dm <sup>3</sup>	64,5	77,2	80	102,6	115,8
Combustion chamber length	mm	288	326	322	331	327
Water content		7	7	7	8	10
Number of elements		5	7	8	8	10
Load loss on flue gas side	Pa	38	40	46	50	53
Load loss on water side 10K J1	Pa	800	1000	2200	3000	3000
Furnace pressure	Pa	20	40	48	28	50
Operating pressure (k)	bar	1	1	1	1	1
Boiler water temperature range	°C	25 + 30	60 + 60	60 + 60	60 + 80	60 + 90
Boiler water two-thermostat differential	K	8	8	8	8	8
Minimum water flow rate of the boiler	kg/h	2300	2600	3200	3800	4300
Maximum diameter of the burner head hole	mm	140	140	140	140	140
Diameter of MB burner fixing holes	mm	170	170	170	170	170
Diameter of flue gas outlet	mm	200	260	260	300	300
Power supply	VHz	230/50	230/50	230/50	230/50	230/50
Mains fuse	A	4	4	4	4	4
Maximum electrical unit power input	W	575	575	575	575	575
Net weight	kg	340	380	420	470	510
CE Certificate	N°	880V0560				

#### NOTE

Boilers in compliance with current standards (CE mark under Directives CEE/73/23, CEE/90/269 and CEE/90/269) must be installed according to the Manufacturer's instructions. The burner must be regulated to give a CO<sub>2</sub> value as shown in the table below.

FUEL	%	MODEL				
		80	100	120	140	160
Natural gas G 20	% CO <sub>2</sub>	9,1 + 0,7	9,1 + 0,7	9,7 + 0,9	9,8 + 1,0	10,2 + 10,4
Oil	% CO <sub>2</sub>	12,3 + 1,3	12,0 + 1,2	12,4 + 1,3	12,8 + 1,4	12,8 + 1,3

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3.3 Table 2-6

**Table A-6** - VALVE LOSSES IN EQUIVALENT FEET OF PIPE\*  
Screwed, Welded, Flanged, and Flowed Connections

NOMINAL PIPE OR TUBE SIZE (in.)	GLOBE†	90°-Y	45°-Y	ANGLE†	GATE††	SWING CHECK‡	LIFT CHECK
1/2	17	6	6	4	0.8	2	
3/4	18	6	7	7	0.7	4	
1	20	11	9	9	0.9	4	
1 1/4	29	15	12	12	1.0	10	Globe & Vertical Lift Same as Globe Valve**
1 1/2	35	20	13	13	1.1	14	
2	45	24	15	15	1.2	16	
2 1/2	58	30	14	14	1.3	20	
3	69	33	16	16	1.4	25	
3 1/2	84	45	18	18	1.5	30	
4	100	50	21	21	1.6	35	
4 1/2	120	58	22	22	1.7	40	
5	140	71	25	25	1.8	50	
6	170	88	27	27	1.9	60	
8	220	115	33	33	2.1	80	
10	280	148	40	40	2.3	100	
12	330	185	48	48	2.5	120	Angle Lift Same as Globe Valve
14	380	225	58	58	2.7	150	
16	440	270	68	68	2.9	200	
18	500	320	80	80	3.1	240	
20	570	380	95	95	3.3	300	
24	670	460	115	115	3.6	360	

\*Values are for all valves in fully open position.  
 †These losses do not apply to valves with needle seat-type seats.  
 ‡These also apply to the inline ball type check valve.  
 \*\*For Y-pattern globe lift check valves with seat approximately equal to the nominal pipe diameter, use values of 90° Y valve for lift.  
 ††Regular and short pattern plug cock valves, when fully open, have same loss as gate valve. For valve losses of short pattern plug cocks above 6 in., check manufacturer.

Table A-7

FITTING LOSSES IN EQUIVALENT FEET OF PIPE  
Screwed, Welded, Flanged, Flared, and Braided Connections

NOMINAL PIPE OR TUBE SIZE (in.)	SMOOTH BEND ELBOWS						SMOOTH BEND TEES				
	90° Std.	90° Long Rad.	45° Street	45° Std.	45° Street	180° Std.	Flow-Through Branch	Straight-Through Flow			
								No. Reduction	Reduced %	Reduced %	
1/8	1.4	0.9	3.3	0.7	1.1	2.3	1.7	0.9	3.2	1.4	
1/4	1.4	1.0	3.3	0.8	1.2	2.3	2.0	1.0	1.4	1.4	
3/8	2.0	1.4	5.1	0.9	1.4	3.2	4.0	1.4	1.7	2.0	
1/2	2.6	1.7	4.1	1.3	2.1	4.1	5.0	1.7	2.2	2.6	
3/4	3.3	2.3	5.6	1.7	3.0	5.6	7.0	2.3	2.7	3.3	
1	4.0	2.6	4.0	2.1	3.4	5.5	8.5	2.6	2.7	4.0	
1 1/4	5.0	3.3	8.3	2.4	4.5	8.3	10	3.3	4.7	5.0	
1 1/2	6.0	4.1	10	3.3	5.3	10	12	4.1	5.6	6.0	
2	7.3	5.0	12	4.0	6.4	12	15	5.0	7.0	7.3	
2 1/2	9.0	5.8	13	4.7	7.3	13	18	5.8	8.0	9.0	
3	10	6.7	17	5.3	8.5	17	21	6.7	9.0	10	
4	12	8.3	21	6.3	11	21	28	8.3	10	12	
5	13	9	21	7.3	13	23	30	10	14	14	
6	14	10	—	—	—	—	33	10	18	20	
8	20	13	—	—	—	—	40	16	23	25	
10	23	14	—	—	—	—	42	16	23	25	
12	25	15	—	—	—	—	50	19	26	30	
14	24	15	—	—	—	—	55	20	30	34	
16	28	16	—	—	—	—	63	24	32	35	
18	42	20	—	—	—	—	70	28	40	42	
20	50	23	—	—	—	—	81	33	44	50	
24	60	30	—	—	—	—	94	40	50	60	

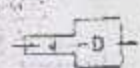
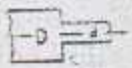




NOMINAL PIPE OR TUBE SIZE (in.)	MITRE ELBOWS			
	90° El	90° El	45° El	30° El
1/8	2.7	1.1	0.4	0.3
1/4	3.0	1.2	0.7	0.4
3/8	4.0	1.5	0.8	0.5
1/2	5.0	2.1	1.0	0.7
3/4	7.0	3.0	1.4	0.9
1	8.0	3.4	1.5	1.1
1 1/4	10	4.5	2.3	1.3
1 1/2	11	5.2	3.0	1.7
2	13	6.4	3.3	2.0
2 1/2	18	7.3	4.0	2.1
3	21	8.5	4.5	2.7
4	23	11	6.0	3.3
5	25	12	7.0	4.0
6	26	13	8.0	4.1
8	30	17	10	5.0
10	32	21	12	6.0
12	40	23	13	8.0
14	48	25	15	9.0
16	58	31	17	10
18	63	37	19	11
20	70	41	20	12
24	81	48	23	16

180° approximately equal to 1, 180° approximately equal to 1.5

Source: Ref. 7-1 (4)

Table A-2

SPECIAL FITTING LOSSES IN EQUIVALENT FEET OF PIPE

NOM. PIPE OR GUESS SIZE (In.)	SUDDEN ENLARGEMENT* d/D			SUDDEN CONTRACTION* d/D			SHARP EDGE*		PIPE PROJECTION*	
	1/4	1/2	3/4	1/4	1/2	3/4	Entrance	Exit	Entrance	Exit
										
3/8	1.4	0.8	0.2	0.7	0.2	0.3	1.2	.8	1.0	1.1
1/2	1.8	1.1	0.4	0.9	0.3	0.4	1.4	1.2	1.3	1.5
3/4	2.2	1.5	0.6	1.2	0.4	0.5	1.7	1.4	1.6	1.9
1	2.6	2.0	0.7	1.4	0.5	0.7	2.0	1.6	1.7	2.1
1 1/4	3.2	2.4	1.0	1.7	0.7	1.0	2.4	1.9	2.1	2.5
1 1/2	3.8	2.8	1.2	2.0	0.8	1.2	2.8	2.2	2.4	3.0
2	4.6	3.4	1.5	2.4	1.0	1.5	3.4	2.6	2.8	3.5
2 1/2	5.4	4.1	2.0	2.8	1.3	2.0	4.0	3.0	3.2	4.0
3	6.2	4.8	2.5	3.2	1.6	2.5	4.6	3.4	3.6	4.5
3 1/2	7.0	5.5	3.0	3.6	1.9	3.0	5.2	3.8	4.0	5.0
4	7.8	6.2	3.5	4.0	2.2	3.5	5.8	4.2	4.4	5.5
4 1/2	8.6	7.0	4.0	4.4	2.5	4.0	6.4	4.6	4.8	6.0
5	9.4	7.8	4.5	4.8	2.8	4.5	7.0	5.0	5.2	6.5
6	10.2	8.6	5.0	5.2	3.1	5.0	7.6	5.4	5.6	7.0
8	12.2	10.2	6.0	6.2	3.8	6.0	9.2	6.4	6.6	8.0
10	14.2	12.2	7.0	7.4	4.5	7.0	10.8	7.4	7.6	9.0
12	16.2	14.2	8.0	8.4	5.2	8.0	12.4	8.4	8.6	10.0
14	18.2	16.2	9.0	9.4	6.0	9.0	14.0	9.4	9.6	11.0
16	20.2	18.2	10.0	10.4	6.8	10.0	15.6	10.4	10.6	12.0
18	22.2	20.2	11.0	11.4	7.6	11.0	17.2	11.4	11.6	13.0
20	24.2	22.2	12.0	12.4	8.4	12.0	18.8	12.4	12.6	14.0
24	28.2	26.2	14.0	14.4	10.0	14.0	22.4	14.4	14.6	17.0

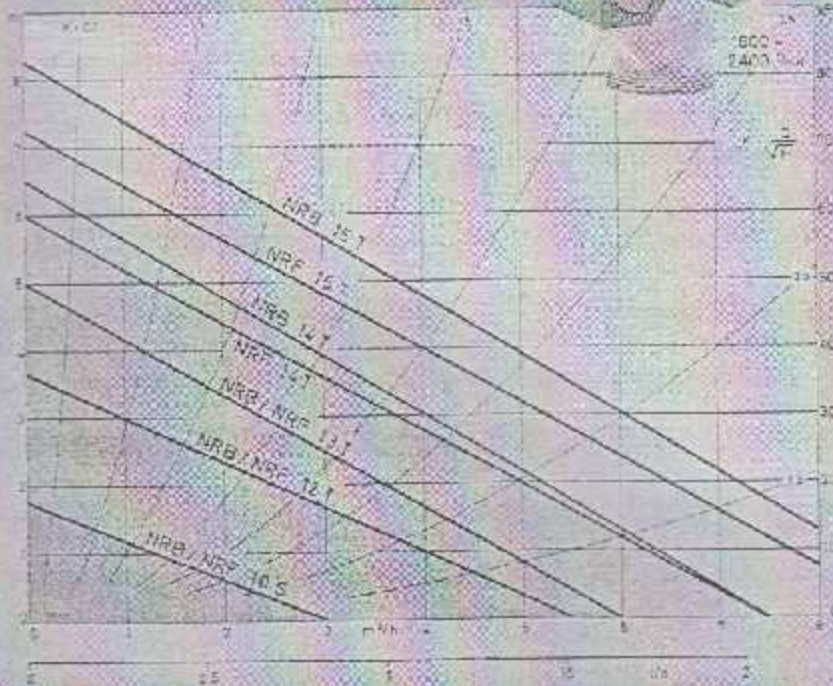
\*Enter table for losses at smallest diameter "d."

Source - Ref. no. (1)

### 3.6 catalogue of BIRALL PUMP



Medium-speed circulators  
with 3 speeds for heating  
and air conditioning



Series:

**NRB**  
**NRF**

Max. system pressure:  
NRB: 12 bar

Max. system pressure:  
NRF: 6 bar

Perm. water temperature:  
110°C

Pump models with 2 or 3 capacity ranges and high starting torque provide greater efficiency and permit the performance to match the demand. A patented safety circuit ensures that the pump will start without problems at its lowest speed.

The "small coils" from Biral provide the highest dimension in reliability and savings precisely in those areas where the performance of small pumps is needed.

Certified Quality System Meeting the International Standards ISO 9001 / EN 29001  
Biral for Quality Pumps!

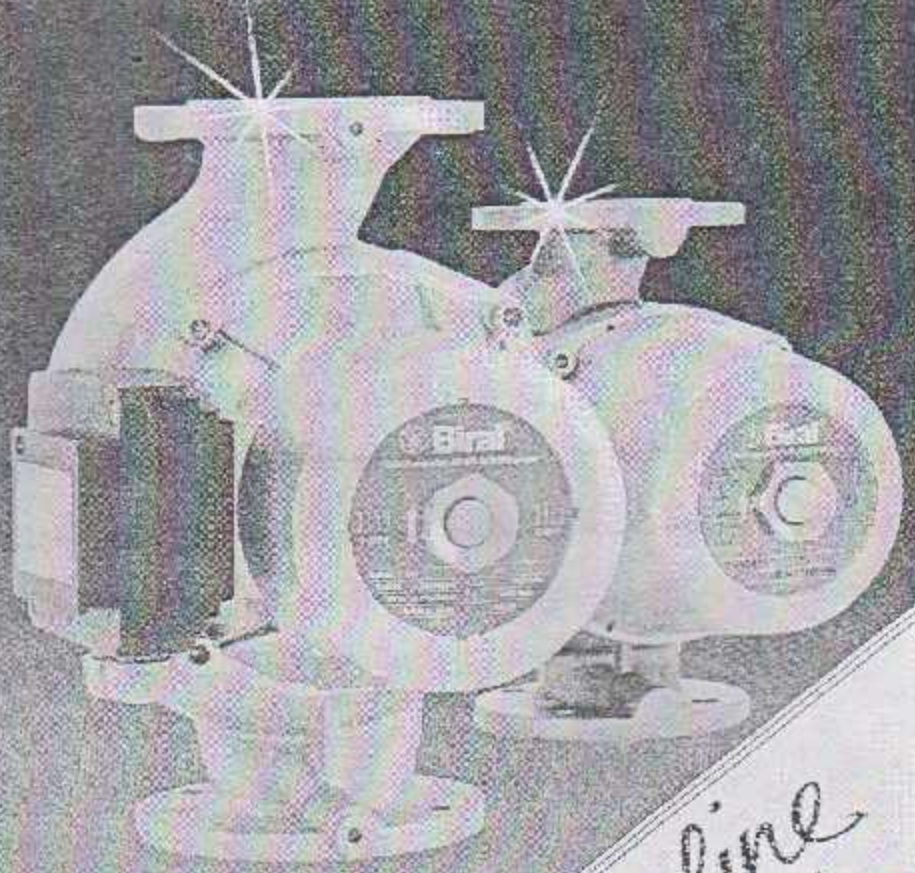


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*The most economical solution in the long run*



*Evoline*  
 The circulators for  
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Table 4.1 Water quality issues with and without private water

Issue	With private water	Without private water
Water quality	Good	Good
Water quantity	Good	Good
Water safety	Good	Good
Water security	Good	Good
Water affordability	Good	Good
Water availability	Good	Good
Water reliability	Good	Good
Water sustainability	Good	Good
Water quality	Good	Good
Water quantity	Good	Good
Water safety	Good	Good
Water security	Good	Good
Water affordability	Good	Good
Water availability	Good	Good
Water reliability	Good	Good
Water sustainability	Good	Good

# APPENDIX B

**Table 4.1 Water supply fixture unit and fixture branch size**

Fixture <sup>a</sup>	Use	Type of Supply Control	Fixture Units <sup>b</sup>	Min. Size of Fixture Branch <sup>d</sup> in
Bathroom group <sup>c</sup>	Private	Flushometer	3	—
Bathroom group <sup>c</sup>	Private	Flush tank for closet	6	—
Bathrub	Private	Faucet	2	1/2
Bathrub	General	Faucet	4	1/2
Clothes washer	Private	Faucet	2	1/2
Clothes washer	General	Faucet	4	1/2
Combination fixture	Private	Faucet	3	1/2
Dishwasher <sup>e</sup>	Private	Automatic	1	1/2
Drinking fountain	Offices, etc.	Faucet 3/4 in.	0.25	1/2
Kitchen sink	Private	Faucet	2	1/2
Kitchen sink	General	Faucet	4	1/2
Laundry trays (1-3)	Private	Faucet	3	1/2
Lavatory	Private	Faucet	1	1/2
Lavatory	General	Faucet	2	1/2
Separate shower	Private	Mixing valve	2	1/2
Service sink	General	Faucet	3	1/2
Shower head	Private	Mixing valve	2	1/2
Shower head	General	Mixing valve	4	1/2
Urinal	General	Flushometer	5	1/2
Urinal	General	Flush tank	3	1/2
Water closet	Private	Flushometer	8	1/2
Water closet	Private	Flushometer/tank	5	1/2
Water closet	Private	Flush tank	5	1/2
Water closet	General	Flushometer	10	1/2
Water closet	General	Flushometer/tank	5	1/2
Water closet	General	Flush tank	5	1/2

Water supply outlets not listed above shall be computed at their maximum demand, but in no case less than the following values:

Fixture Branch <sup>a</sup>	Number of Fixture Units	
	Private Use	General Use
1/2	1	2
3/8	2	4
3/4	3	6
1	6	10

<sup>a</sup>For supply outlets likely to impose continuous demands, estimate continuous supply separately and add to total demand for fixtures.

<sup>b</sup>The given weights are for total demand. For fixtures with both hot and cold water supplies, the weights for maximum separate demands may be taken as three-quarters the listed demand for the supply.

<sup>c</sup>A bathroom group for the purposes of this table consists of not more than one water closet, one lavatory, one bathtub, one shower stall or one water closet, two lavatories, one bathtub or one separate shower stall.

<sup>d</sup>Nominal I.D. pipe size.

<sup>e</sup>Some may require larger sizes—see manufacturer's instructions.

<sup>f</sup>Data extracted from Code Table B.5.2.

Source: Reproduced with permission from The National Standard Plumbing Code, published by The National Association of Plumbing Heating Cooling Contractors.

**Table 4.2**

**Minimum Pressure Required by Typical Plumbing Fixtures**

Fixture Type	Minimum Pressure, <i>psi</i>
Sink and tub faucets	8
Shower	8
Water closet—tank flush	6
Flush valve—urinal	15
Flush valve—siphon jet bowl	
floor-mounted	17
wall-mounted	20
Flush valve—blowout bowl	
floor-mounted	20
wall-mounted	25
Garden hose	
1/2-in. sill cock	15
3/4-in. sill cock	30
Drinking fountain	5

Source: EPA Manual of Individual Water Supply Sinks, 1975 and manufacturers' data.

**Recommended Flow Rates for Typical Plumbing Fixtures**

Fixture Type	Flow, <i>gpm</i>
Lavatory	1
Sink	1.5
Bath tub	4
Laundry tray	3
Shower	2-10
Water closets	
tank type	3
flush valve*	15-40
Urinal flush valve	15
Garden hose	
1/2-in. sill cock	2 1/2
3/4-in. sill cock	5
Drinking fountain	1/4

Source: Data extracted from various sources.

\*Wide range of flows; depends on flow pressure.

**Table for Estimating Demand**

Supply Systems Predominantly for Flush Tanks		Supply Systems Predominantly for Flushometers	
Load, WSFU*	Demand, <i>gpm</i>	Load, WSFU*	Demand, <i>gpm</i>
6	5	—	—
10	8	10	27
15	11	15	31
20	14	20	35
25	17	25	38
30	20	30	41
40	25	40	47
50	29	50	51
60	33	60	55
80	39	80	62
100	44	100	68
120	49	120	73
140	53	140	78
160	57	160	83
180	61	180	87
200	65	200	91
225	70	225	96
250	75	250	100
300	83	300	110
400	103	400	125
500	125	500	140
750	170	750	175
1000	210	1000	210
1250	240	1250	240
1500	270	1500	270
1750	300	1750	300
2000	325	2000	325
2500	380	2500	380
3000	435	3000	435
4000	525	4000	525
5000	600	5000	600
6000	670	6000	670
7000	730	7000	730
8000	780	8000	780
9000	830	9000	830
10,000	870	10,000	870

\*Water Supply Fixture Units

Source: Reproduced with permission from The National Standard Plumbing Code, published by The National Association of Plumbing Heating Cooling Contractors.

**Table 4.6 Horizontal Fixture branches and stacks**

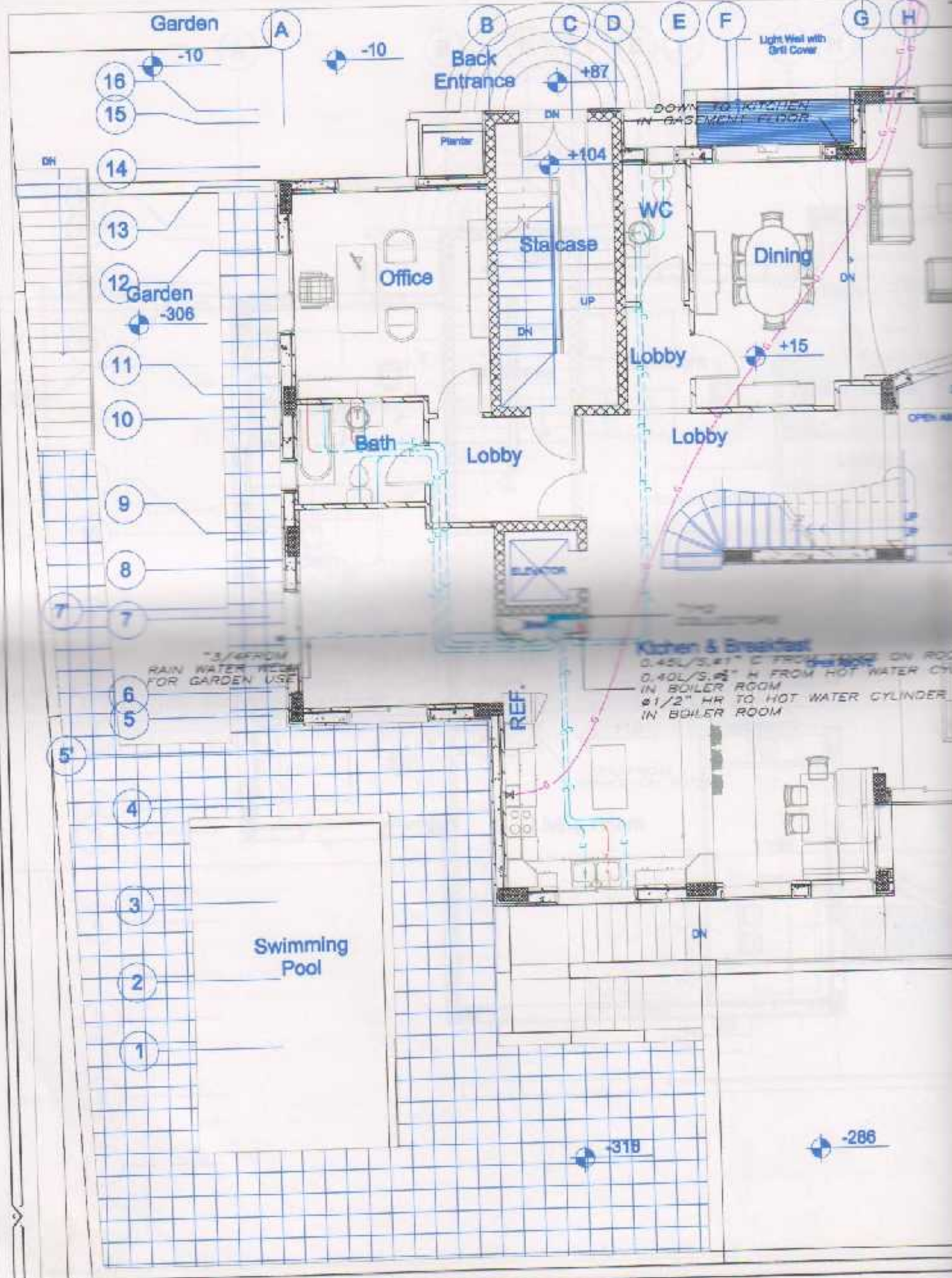
Diameter of Pipe, in.	Maximum Number of Fixture Units That May Be Connected to			
	Any Horizontal Fixture Branch, <sup>a</sup> dfu	For Stack, One Stack of Three Branch Intervals or Less, dfu	Stacks with More Than Three Branch Intervals - Total for Stack, dfu	Total at One Branch Interval, dfu
1 1/2	5	4	8	2
2	8	10	24	6
2 1/2	12	20	42	9
3	20 <sup>b</sup>	48 <sup>b</sup>	72 <sup>b</sup>	20 <sup>b</sup>
4	160	240	500	90
5	360	540	1100	200
6	620	960	1900	350
8	1400	2200	3600	600
10	2500	3800	5600	1000
12	3500	6000	8400	1500
15	7000			

<sup>a</sup>Does not include branches of the building drain.

<sup>b</sup>Not more than two water closets or bathroom groups within each branch interval nor more than six water closets or bathroom groups on the stack.

Note: Stacks shall be sized according to the total accumulated connected load at each story or branch interval and may be reduced in size as this load decreases to a minimum diameter of half of the largest size required.

Source: Reprinted with permission of the National Standard Plumbing Code, published by The National Association of Plumbing Heating Cooling Contractors.



16  
15

14

13

12

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8

7

7

6

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3

2

1

Swimming Pool

Office

Bath

Lobby

Staircase

Lobby

Lobby

Dining

Back Entrance

Garden

Garden

**Kitchen & Breakfast**  
 0.45L/S, 81" C FROM TAP ON ROOF  
 0.40L/S, 82" H FROM HOT WATER CYL. IN BOILER ROOM  
 8 1/2" HR TO HOT WATER CYLINDER IN BOILER ROOM

3/4" FROM RAIN WATER WELLS FOR GARDEN USE

DOWN TO KITCHEN IN BASEMENT FLOOR

Light Well with Grill Cover

Planter

WC

UP

DN

DN

DN

ELEVATOR

REF.

DN

-318

-286

-10

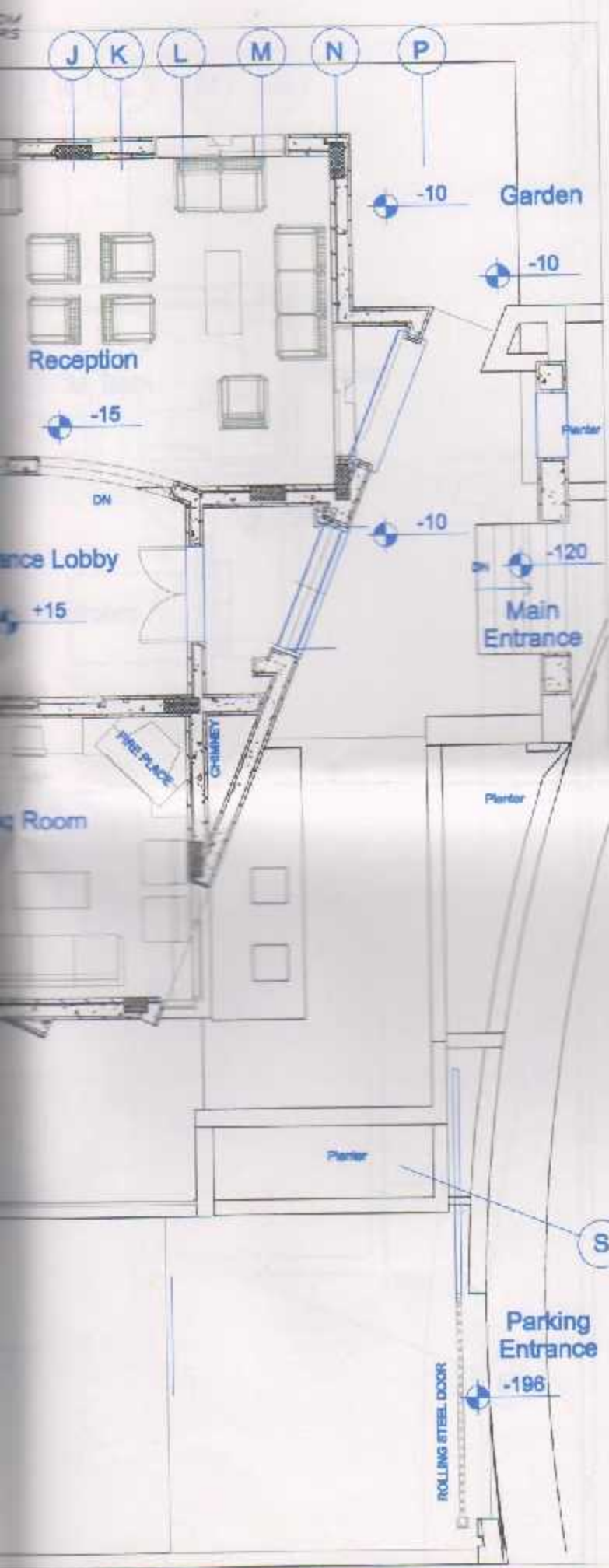
+87

+104










+15

-10

-308



# LEGEND

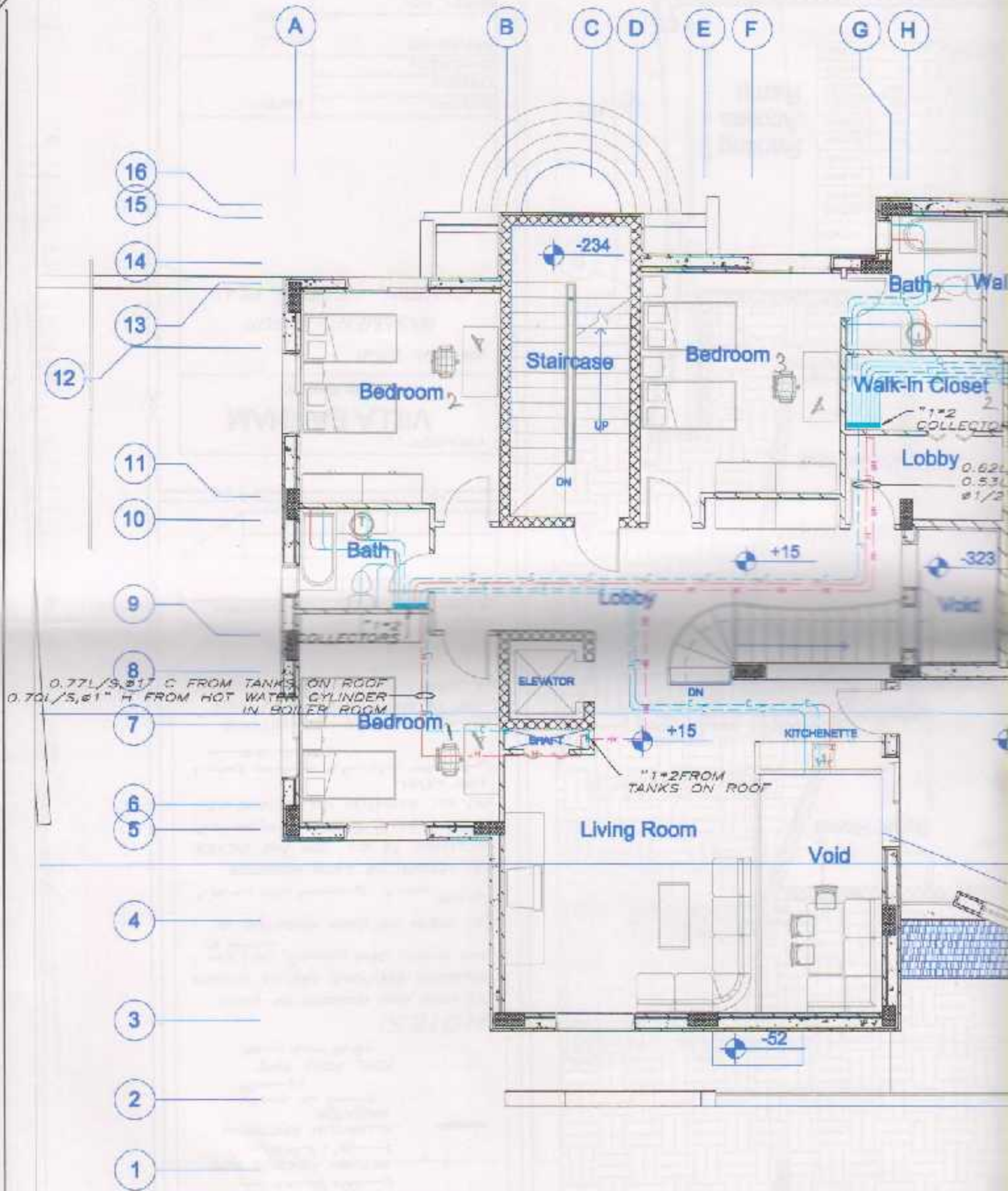
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صمام إغلاق
-  FLOAT VALVE  
صمام عوامية
-  HOT WATER G.S. PIPE  
مسانورة ماء ساخن فولاذية
-  COLD WATER G.S. PIPE  
مسانورة ماء بارد فولاذية
-  HOT WATER POLYETHYLENE PIPE  
أنبوب ماء ساخن من البولييثيلين
-  COLD WATER POLYETHYLENE PIPE  
أنبوب ماء بارد من البولييثيلين
-  GAS SOFT COPPER PIPE  
أنبوب نحاسي للاستعمال الغاز
-  HOT WATER RETURN PIPE  
أنبوب راجع للمياه الساخنة
-  SOLAR SYSTEM PIPE  
مواسير النظام الشمسي

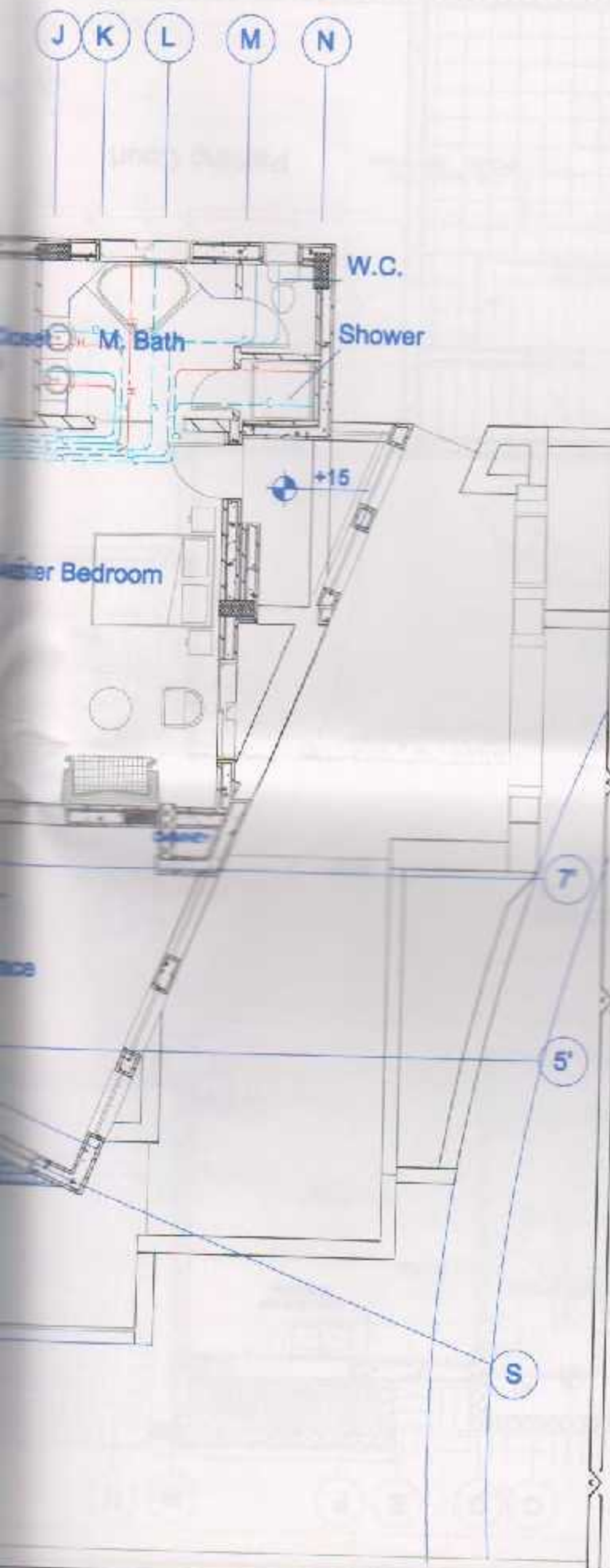
R	REV. BY	DATE	DESCRIPTION

PROJECT:  
**VILLA BARHAM**  
Treh, Ramallah







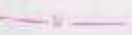
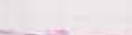
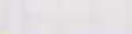
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**GROUND FLOOR  
GAS & WATER PIPING PLAN**

DESIGN:	DATE:
DRAWN:	SCALE: 1/100
CHECKED: A. KHAFAJAH	
APPROVED :	SHEET NO: <b>M5</b>





# LEGEND

-  SHUT-OFF VALVE  
صمام اغلاق
-  FLOAT VALVE  
صواملة
-  HOT WATER G.S. PIPE  
مانورة ماء ساخن فولاذية
-  COLD WATER G.S. PIPE  
مانورة ماء بارد فولاذية
-  HOT WATER POLYETHYLENE PIPE  
انبوب ماء ساخن من البوليثلين
-  COLD WATER POLYETHYLENE PIPE  
انبوب ماء بارد من البوليثلين
-  GAS SOFT COPPER PIPE  
انبوب نحاسي لاستعمال الغاز
-  HOT WATER RETURN PIPE  
انبوب راجع للحرارة السائلة
-  SOLAR SYSTEM PIPE  
مواسير النظام الشمسي

R	REV. BY	DATE	DESCRIPTION

PROJECT:  
**VILLA BARHAM**  
Tireh, Ramallah

DRAWING TITLE:  
  
FIRST FLOOR  
WATER PIPING PLAN

DESIGN:	DATE:
DRAWN:	
CHECKED:	
APPROVED :	SCALE: 1/100
SHEET NO: <b>M6</b>	







16

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1

A

B

C

D

E

F

G

H

Garden

W.C.

Bath

Staircase

Light Well with Grill Cover

Maid

Kitchen

COLLECTORS

Bath

Guest Bedroom

ELEVATOR

Multi Purpose Hall

Swimming Pool

Parking Court

3/4" x 2" FOR POOL HEATING

8

7

6

5

4

1

2

+0

+0

9

3

W D W

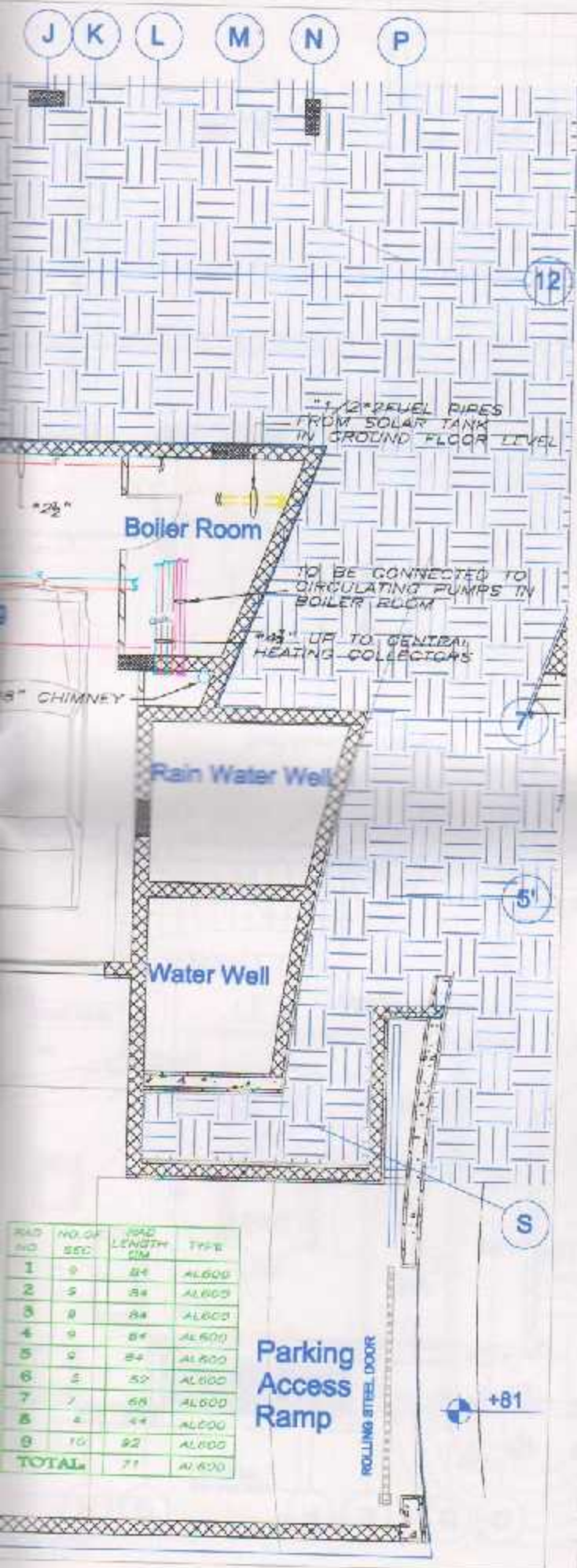
+0

UP

UP

DOWN

UP



## LEGEND

- FLOW HEATING PIPE  
خط تدفئة أحمر
- RETURN HEATING PIPE  
خط تدفئة أزرق
- ▭ ALUMINUM SECTIONAL RADIATOR  
رادياتور من ألومنيوم
- FUEL FEED PIPE  
أنبوب تزويد للوقود

### NOTES:

- (1) KIND AND NUMBER OF EACH RADIATOR SECTIONS ARE AS SHOWN.  
النوع وعدد الأقسام لكل رادياتور كما هو موضح
- (2) WIDTH OF EACH RADIATOR IS 8 CM.  
عرض كل رادياتور 8 سم
- (3) LENGTH OF EACH RADIATOR INCLUDES 12 CM. FOR THE VALVES.  
كل قسم طول الرادياتور 12 سم للصمامات
- (4) ALL RADIATOR VALVES ARE AT LOW LEVEL.  
جميع صمامات الرادياتور تكون في مستوى منخفض
- (5) ALL RADIATOR CONNECTIONS ARE 16 MM. POLYETHYLENE PIPE.  
كل التوصيلات للرادياتور تكون من البوليإيثيلين قطر 16 ملم

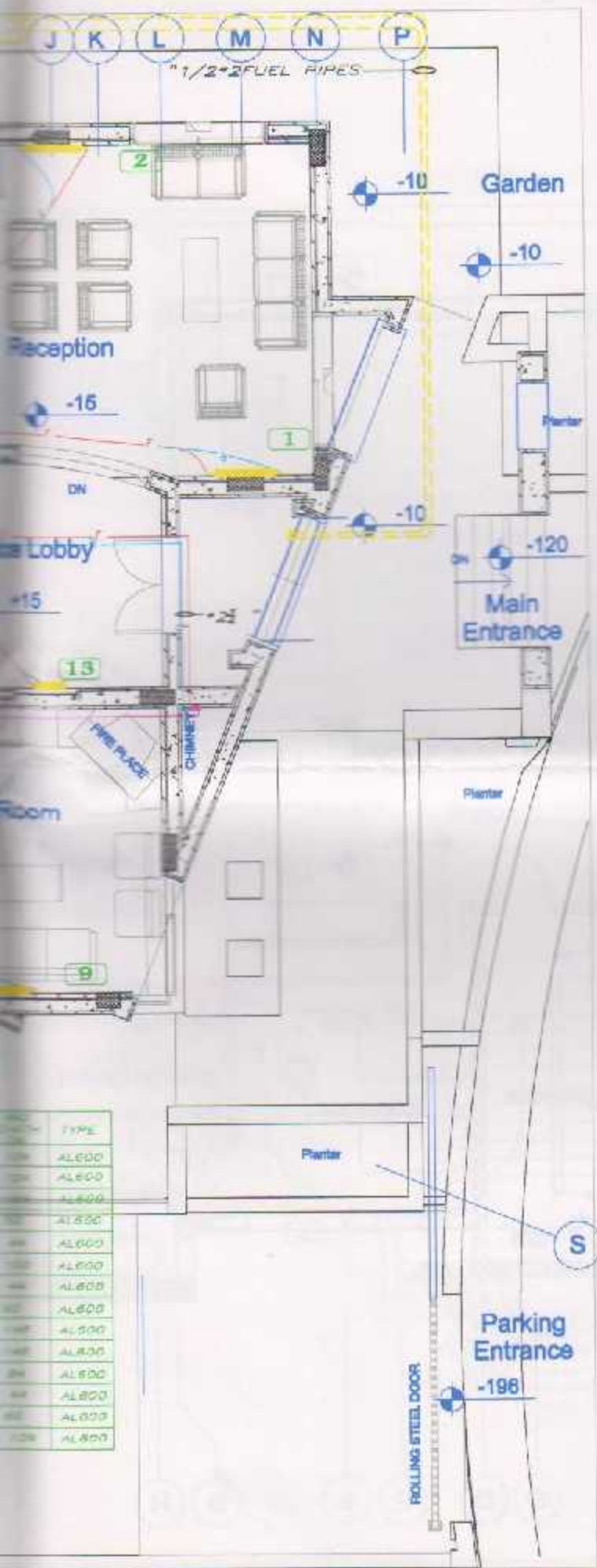
R	REV. BY	DATE	DESCRIPTION

PROJECT:  
**VILLA BARHAM**  
Treh, Ramallah

DRAWING TITLE:  
**BASEMENT FLOOR  
CENTRAL HEATING PLAN**

DESIGN:	DATE:
DRAWN:	
CHECKED:	
APPROVED :	SCALE: 1/100
SHEET NO: <b>M7</b>	

RAD NO	NO. OF SEC.	RAD LENGTH CM	TYPE
1	9	84	AL600
2	5	84	AL600
3	8	84	AL600
4	9	84	AL600
5	9	84	AL600
6	5	50	AL600
7	7	68	AL600
8	4	44	AL600
9	10	92	AL600
<b>TOTAL</b>	<b>71</b>	<b>AL600</b>	



# LEGEND

- FLOW HEATING PIPE  
خط تدفئة داخلي
- RETURN HEATING PIPE  
خط تدفئة خارجي
- ▬ ALUMINUM SECTIONAL RADIATOR  
راديوسر من الالمنيوم
- FUEL FEED PIPE  
انيوبيا تزويد للوقود

## NOTES:

- (1) KIND AND NUMBER OF EACH RADIATOR SECTIONS ARE AS SHOWN.  
ان نوع وعدد الاقسام لكل راديوسر كما هو مبين
- (2) WIDTH OF EACH RADIATOR IS 8 CM.  
عرض كل اقسام 8 سم
- (3) LENGTH OF EACH RADIATOR INCLUDES 12 CM FOR THE VALVES.  
يشمل طول الراديوسر 12 سم للامسامات
- (4) ALL RADIATOR VALVES ARE AT LOW LEVEL.  
جميع صمامات الراديوسرات تكون في مستوى منخفض
- (5) ALL RADIATOR CONNECTIONS ARE 16 MM POLYETHYLENE PIPE.  
كل التوصيلات للراديوسرات تكون من البوليثيلين قطر 16 ملم

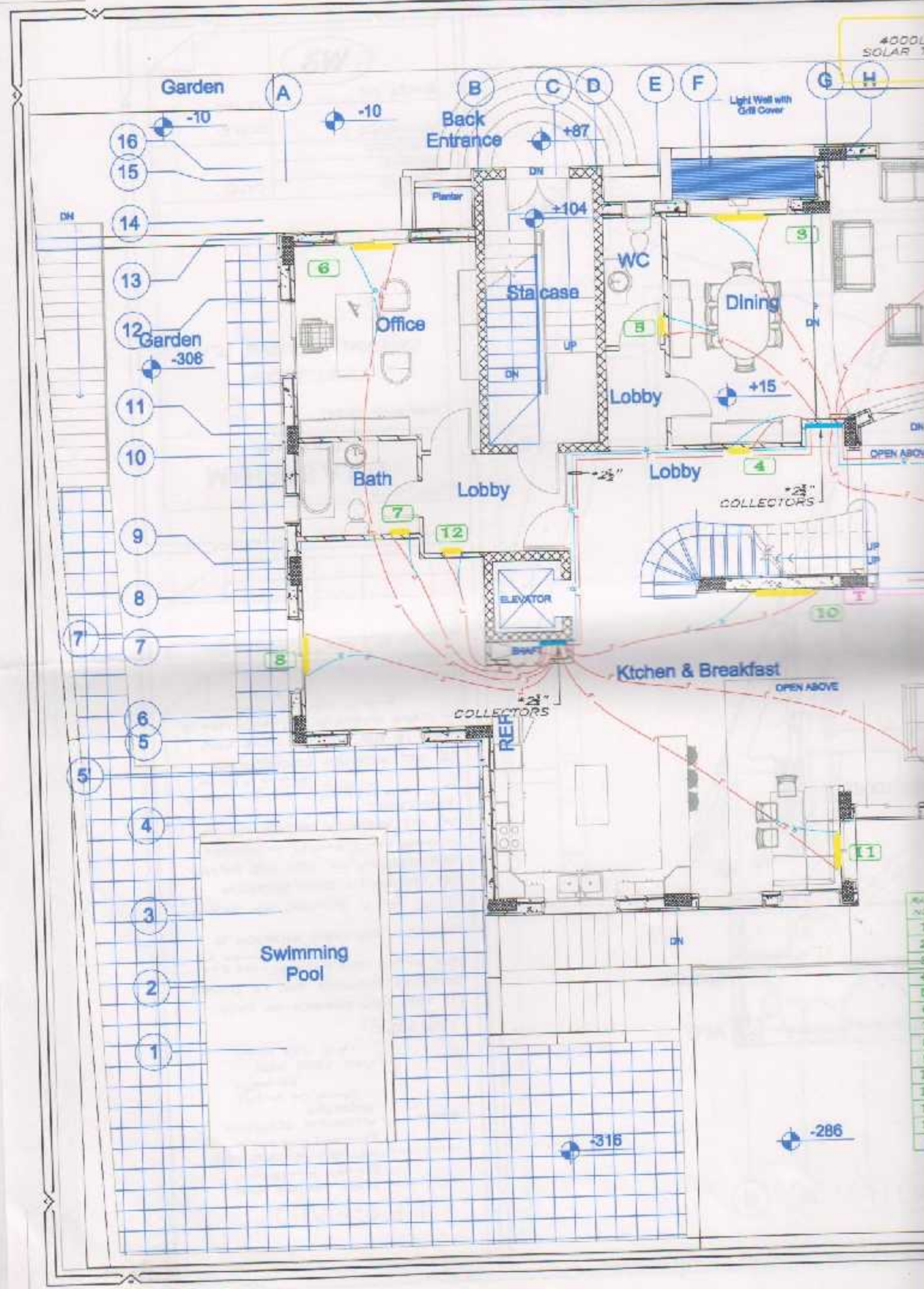
R	REV.	BY	DATE	DESCRIPTION

PROJECT:  
**VILLA BARHAM**  
Tlrah, Ramallah

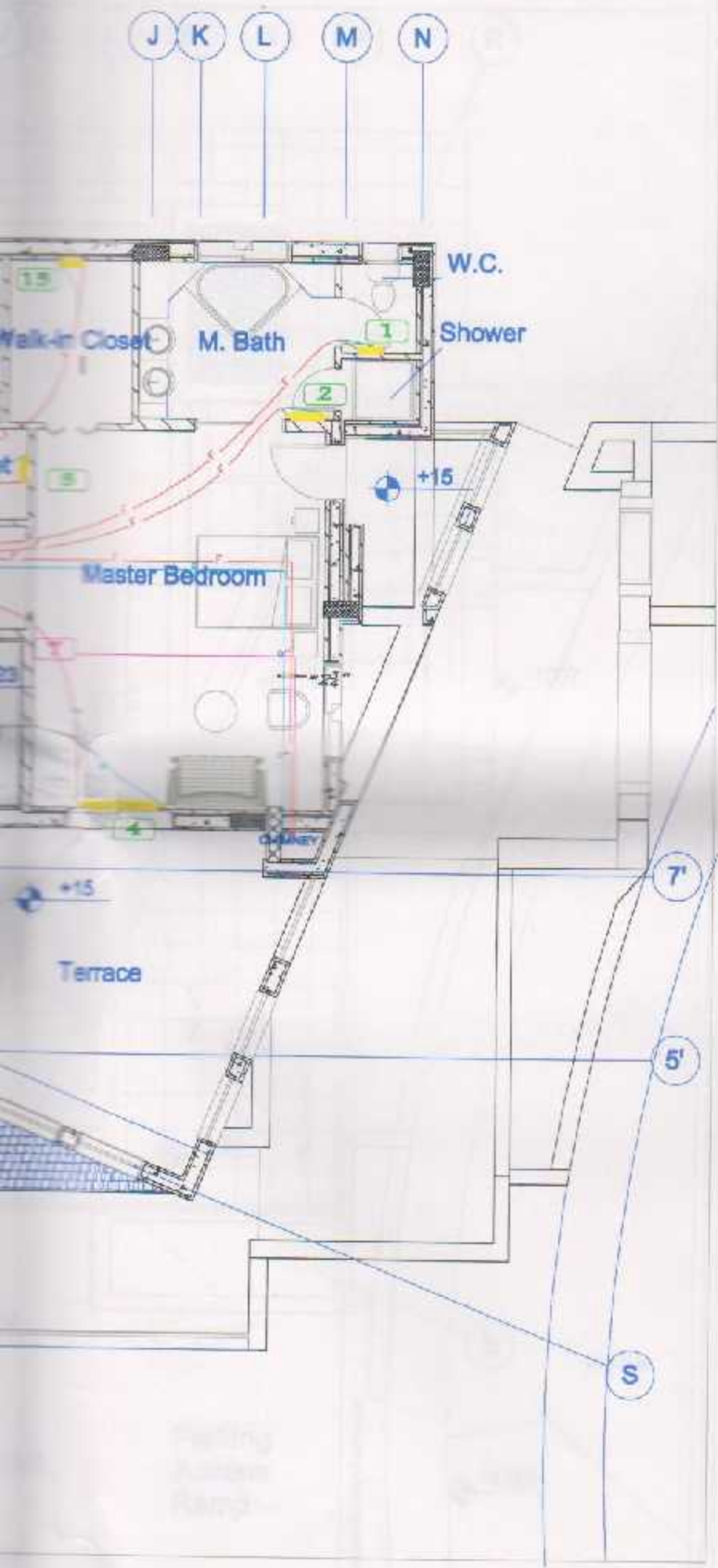
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**GROUND FLOOR  
CENTRAL HEATING PLAN**

DESIGN:	DATE:
DRAWN:	SCALE: 1/100
CHECKED:	
APPROVED :	
SHEET NO:	<b>M8</b>

4000L SOLAR T...



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15
16



## LEGEND

- FLOW HEATING PIPE  
خط تدفئة داخلي
- RETURN HEATING PIPE  
خط تدفئة خارجي
- ALUMINUM SECTIONAL RADIATOR  
رادياتر من ألومنيوم
- FUEL FEED PIPE  
أنبوب تزويد للوقود

### NOTES:

- (1) KIND AND NUMBER OF EACH RADIATOR SECTIONS ARE AS SHOWN.  
النوع وعدد الأقسام لكل رادياتر كما هو مبين
- (2) WIDTH OF EACH RADIATOR IS 8 CM.  
عرض كل رادياتر 8 سم
- (3) LENGTH OF EACH RADIATOR INCLUDES 12 CM. FOR THE VALVES.  
يشتمل طول الرادياتر 12 سم للضمامات
- (4) ALL RADIATOR VALVES ARE AT LOW LEVEL.  
جميع ضمامات الرادياترات تكون في مستوى منخفض
- (5) ALL RADIATOR CONNECTIONS ARE 16 MM. POLYETHYLENE PIPE.  
كل التوصيلات للرادياترات تكون من البوليثلين قطر 16 مم

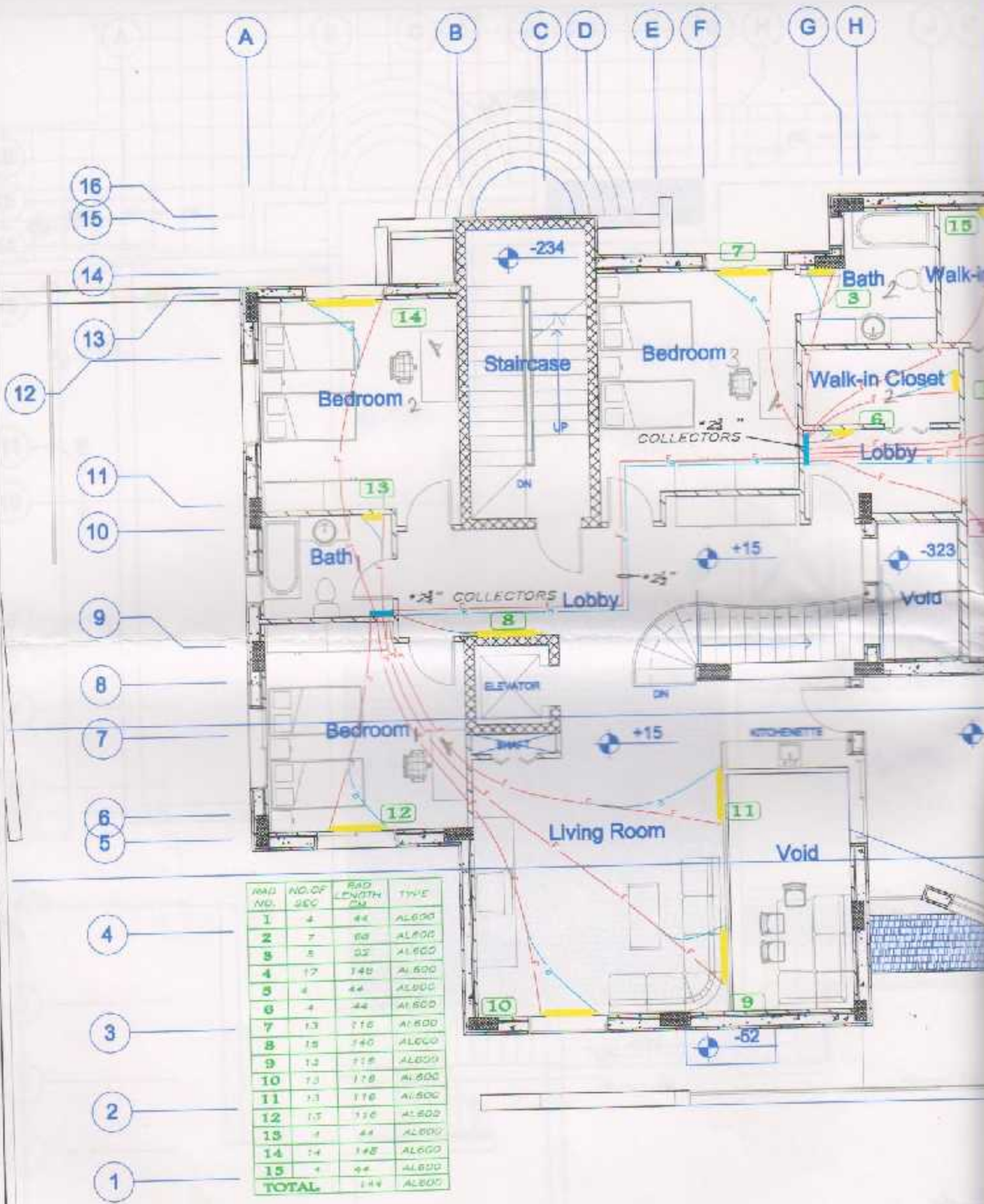
REV.	BY	DATE	DESCRIPTION

PROJECT:  
**VILLA BARHAM**  
Tireh, Ramallah

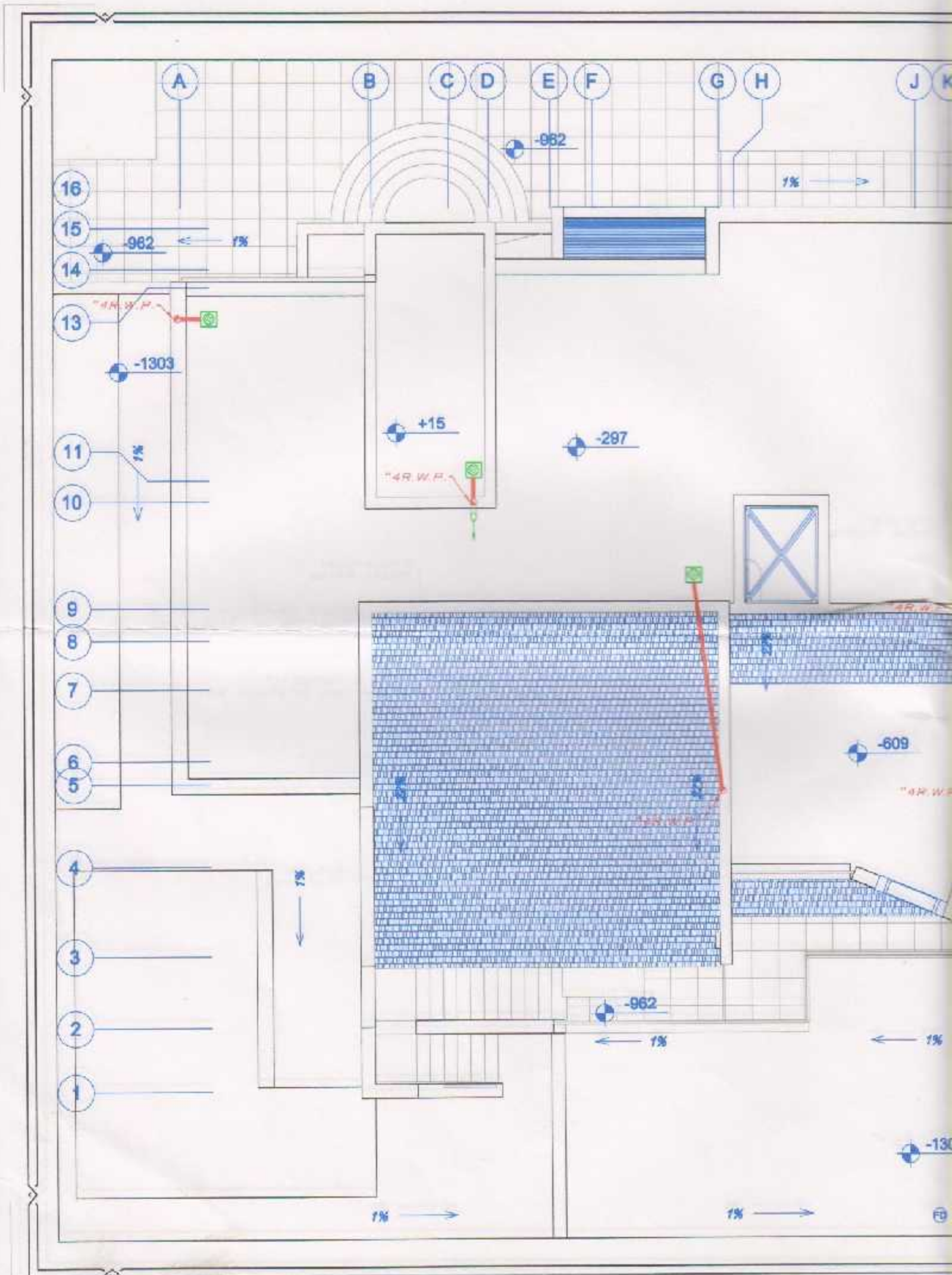
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**FIRST FLOOR  
CENTRAL HEATING PLAN**

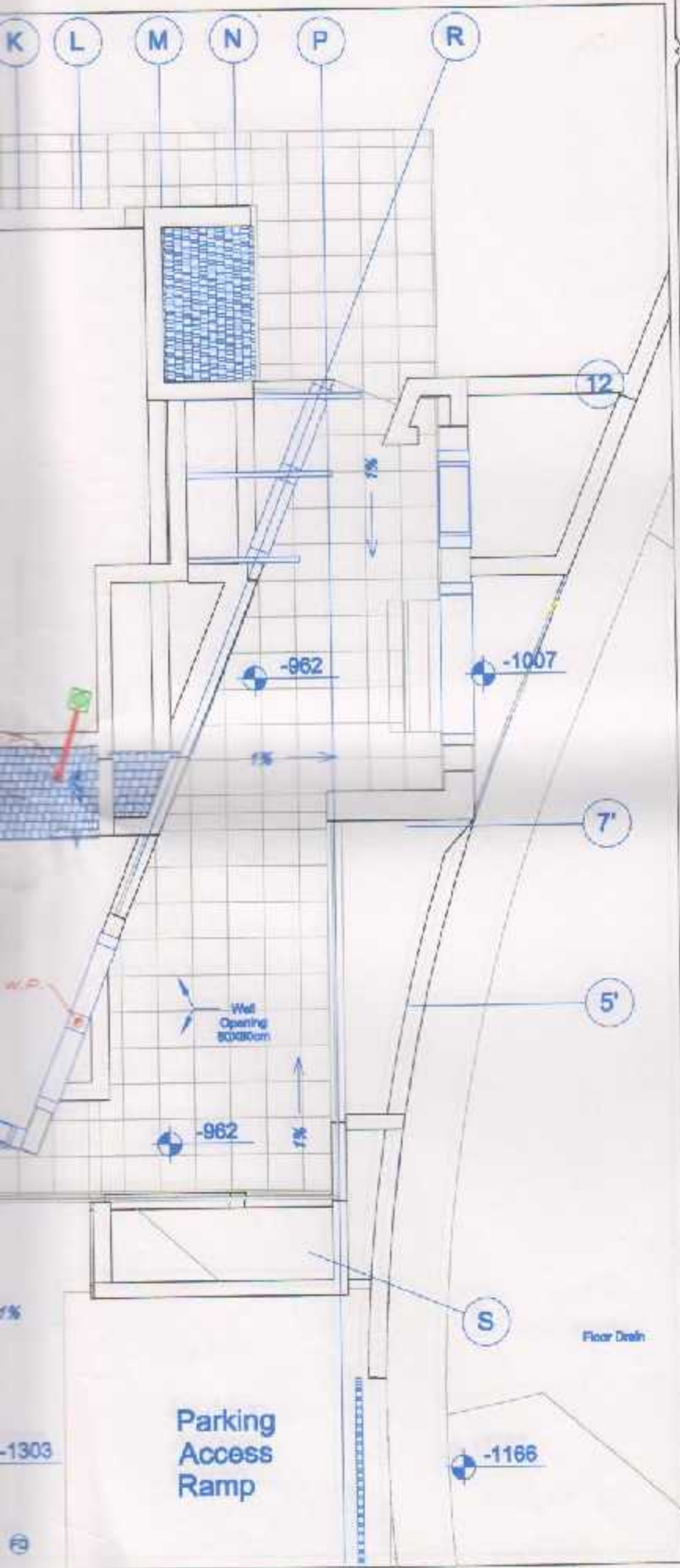
DESIGN:	DATE:
DRAWN:	
CHECKED:	
APPROVED :	SCALE:
SHEET NO:	1/100

**M9**



RAD NO.	NO. OF SEC	RAD LENGTH CM	TYPE
1	4	44	AL600
2	7	68	AL600
3	5	32	AL600
4	17	148	AL600
5	4	44	AL600
6	4	44	AL600
7	13	116	AL600
8	15	140	AL600
9	12	118	AL600
10	10	118	AL600
11	10	116	AL600
12	10	116	AL600
13	4	44	AL600
14	14	148	AL600
15	4	44	AL600
<b>TOTAL</b>	<b>144</b>		<b>AL600</b>





# LEGEND

- 
**3" S2**  
 P.V.C. DRAIN PIPE  
 DIAMETER (3") SLOPE (2%)  
 ماسورة تصريف بلاستيك  
 قطر (3) إنش ميل (2%)
- 
**4" P.V.C. DRAIN PIPE  
(2% SLOPE)**  
 ماسورة تصريف بلاستيك  
 قطر 4 إنش ميل 2%
- 
**2" P.V.C. DRAIN PIPE  
(2% SLOPE)**  
 ماسورة تصريف بلاستيك  
 قطر 2 إنش ميل 2%
- 
**1" FLOOR TRAP**  
 مصيدة روائح 1 إنش
- 
**4 1/2" FLOOR DRAIN**  
 مصرف أرضي 4 1/2 إنش
- 
**4" CLEAN OUT**  
 فتحة تنظيف 4 إنش
- 
**4" RAIN WATER PIPE**  
 ماسورة مطر 4 إنش
- 
**4" RAIN WATER DRAIN**  
 مصرف مطر 4 إنش
- 
**4" RAIN WATER  
(FREE OUTLET)**  
 مخرج حر لماسورة مطر
- 
**4" VENT & RISER**  
 ماسورة تهوية عمودية  
 4 إنش
- 
**CONCRETE MANHOLE**  
 منهل أسمنتي

REV.	REV. BY	DATE	DESCRIPTION

PROJECT:

**VILLA BARHAM**

Tireh, Ramallah

DRAWING TITLE:

TOP ROOF  
DRAIN PLAN

DESIGN:

DATE:

DRAWN:

CHECKED:

APPROVED :

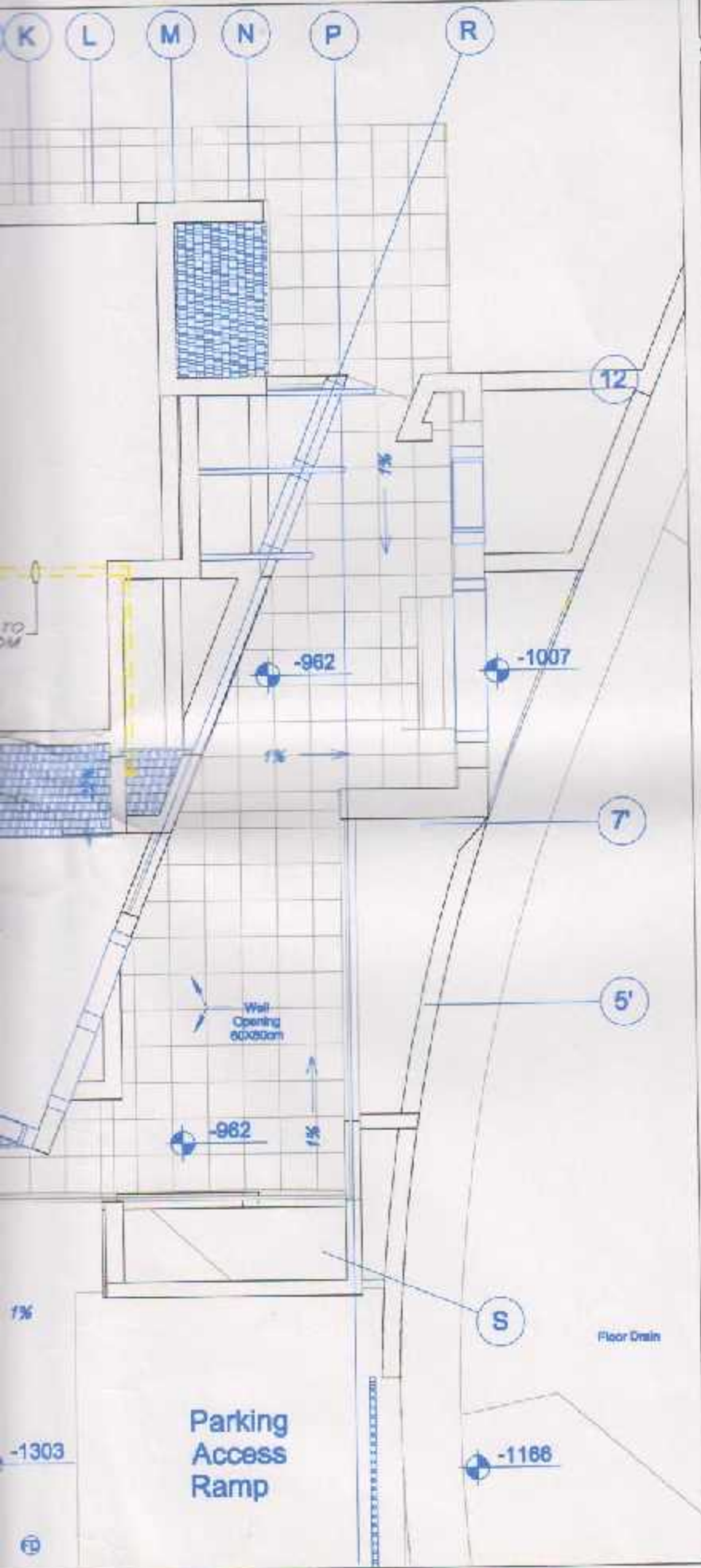
SCALE:

1/100










SHEET NO:

**M10**





# LEGEND

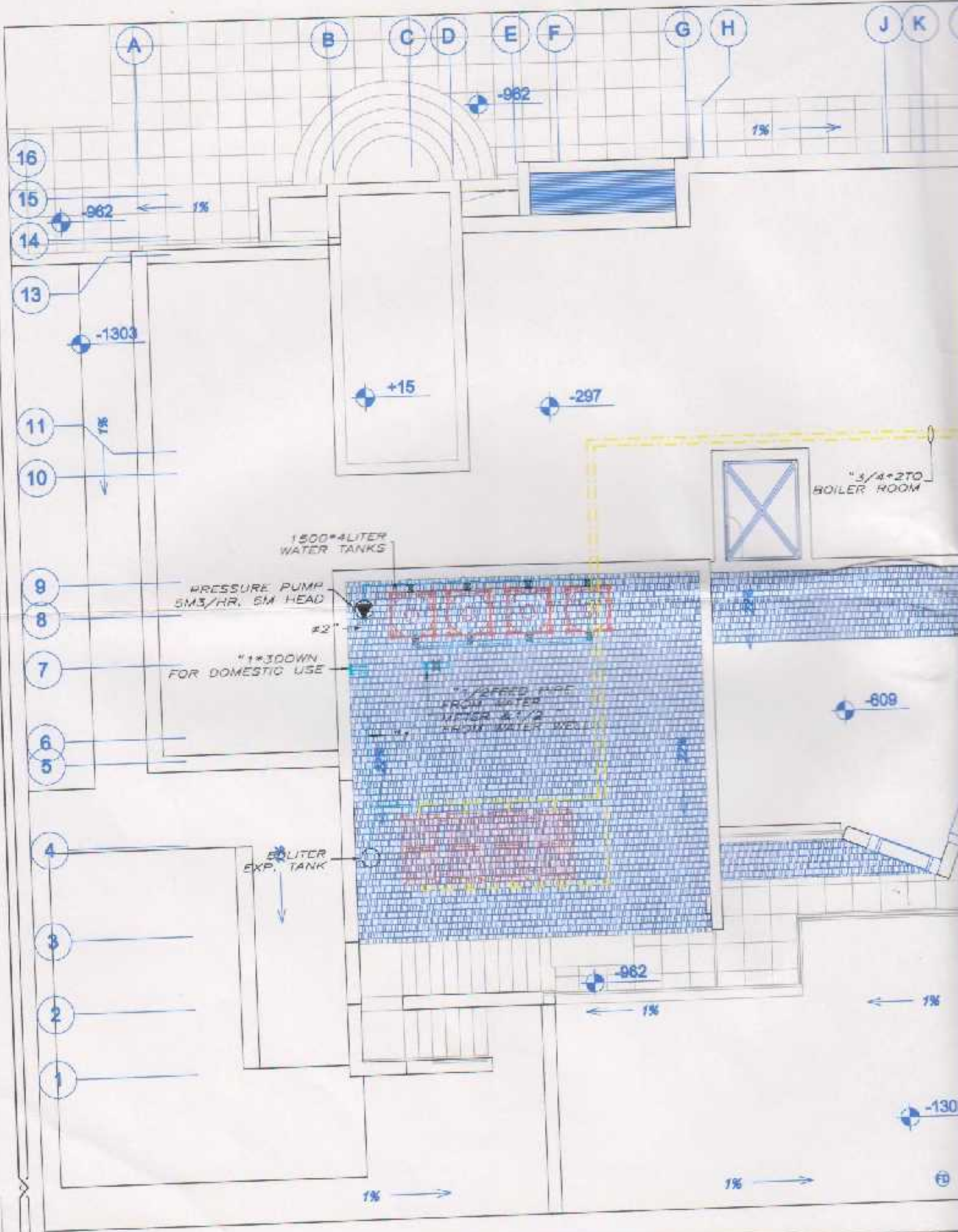
-  SHUT-OFF VALVE  
صمام اضلال
-  FLOAT VALVE  
صمام عائم
-  HOT WATER G.S. PIPE  
ماسورة ماء ساخن فولاذية
-  COLD WATER G.S. PIPE  
ماسورة ماء بارد فولاذية
-  HOT WATER POLYETHYLENE PIPE  
انبوب ماء ساخن من البوليثلين
-  COLD WATER POLYETHYLENE PIPE  
انبوب ماء بارد من البوليثلين
-  GAS SOFT COPPER PIPE  
انبوب نحاسي لامتصاص الغاز
-  HOT WATER RETURN PIPE  
انبوب راجع للمياه الساخنة
-  SOLAR SYSTEM PIPE  
مواسير النظام الشمسي

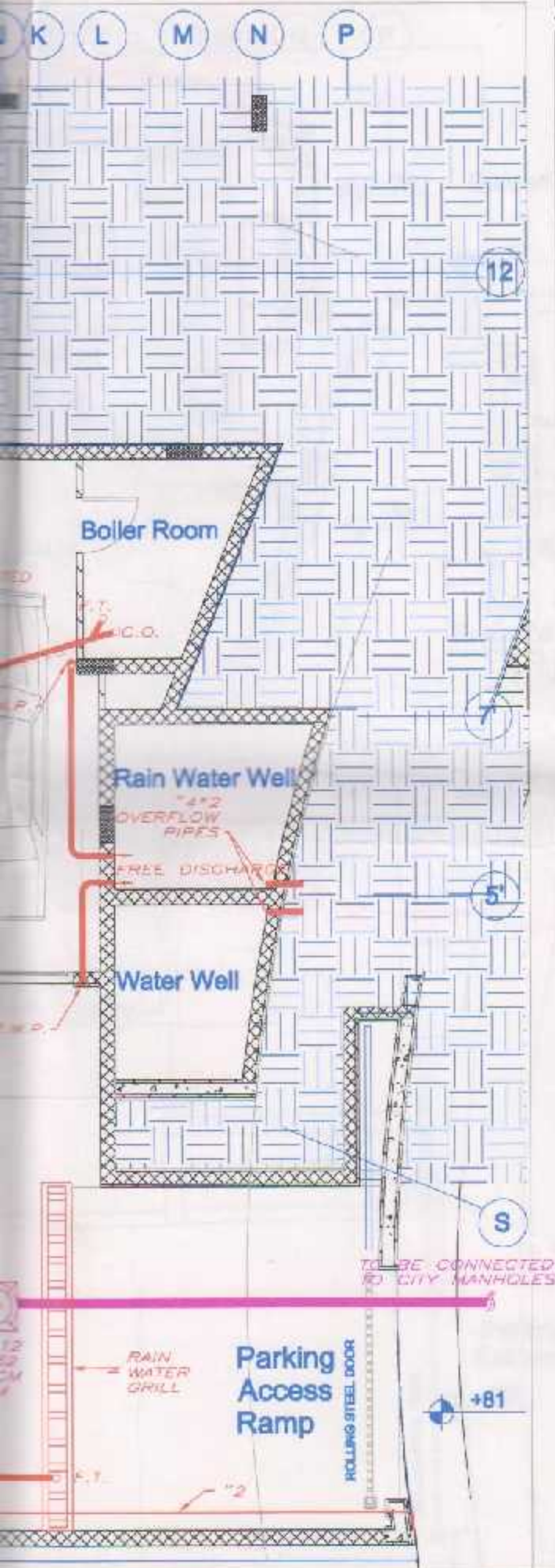
#	REV. BY	DATE	DESCRIPTION

PROJECT:  
**VILLA BARHAM**  
Treh, Ramallah

DRAWING TITLE:  
  
**TOP ROOF  
WATER PIPING PLAN**

DESIGN:	DATE:
DRAWN:	SCALE: 1/100
CHECKED:	
APPROVED :	SHEET NO:
<b>M11</b>	





# LEGEND

-  P.V.C. DRAIN PIPE  
DIAMETER (Ø) SLOPE (S)  
ماسورة تصريف بلاستيك  
قطر (Ø) ميل (S)
-  4" P.V.C. DRAIN PIPE  
(2% SLOPE)  
ماسورة تصريف بلاستيك  
قطر 4 انش ميل 2%
-  2" P.V.C. DRAIN PIPE  
(2% SLOPE)  
ماسورة تصريف بلاستيك  
قطر 2 انش ميل 2%
-  4" FLOOR TRAP  
مصيدة روائح 4 انش
-  4 1/2" FLOOR DRAIN  
مصرف ارضي 4 1/2 انش
-  4" CLEAN OUT  
فتحة تنظيف 4 انش
-  4" RAIN WATER PIPE  
ماسورة مطر 4 انش
-  4" RAIN WATER DRAIN  
مصرف مطر 4 انش
-  4" RAIN WATER (FREE OUTLET)  
مخرج حر لماسورة مطر
-  4" VENT & RISER  
ماسورة تهوية عمودية  
4 انش
-  CONCRETE MANHOLE  
منفذ اسمنتي

R	REV. BY	DATE	DESCRIPTION

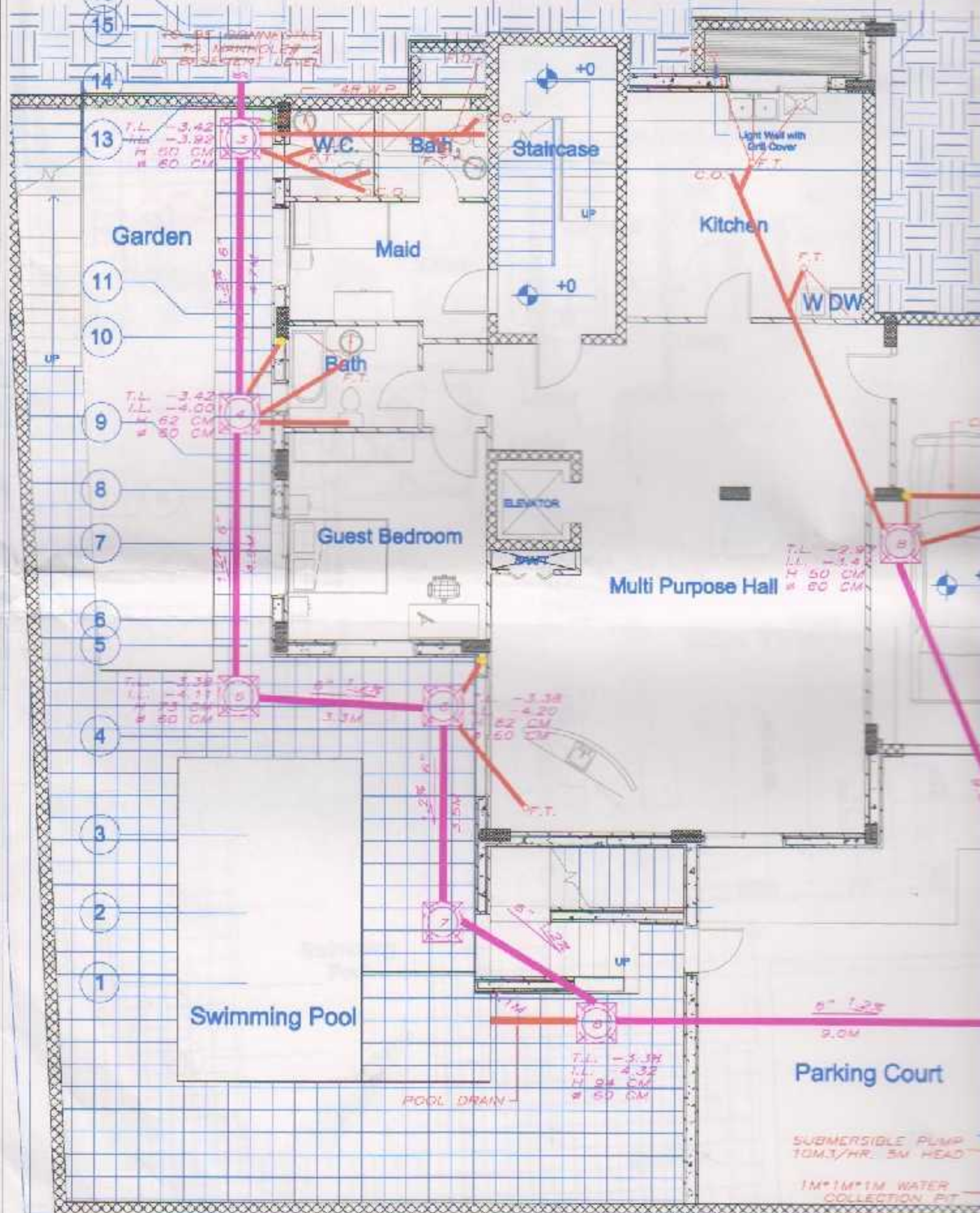
PROJECT:  
**VILLA BARHAM**  
Tirah, Ramallah

DRAWING TITLE:  
**BASEMENT FLOOR  
SANITARY PLAN**

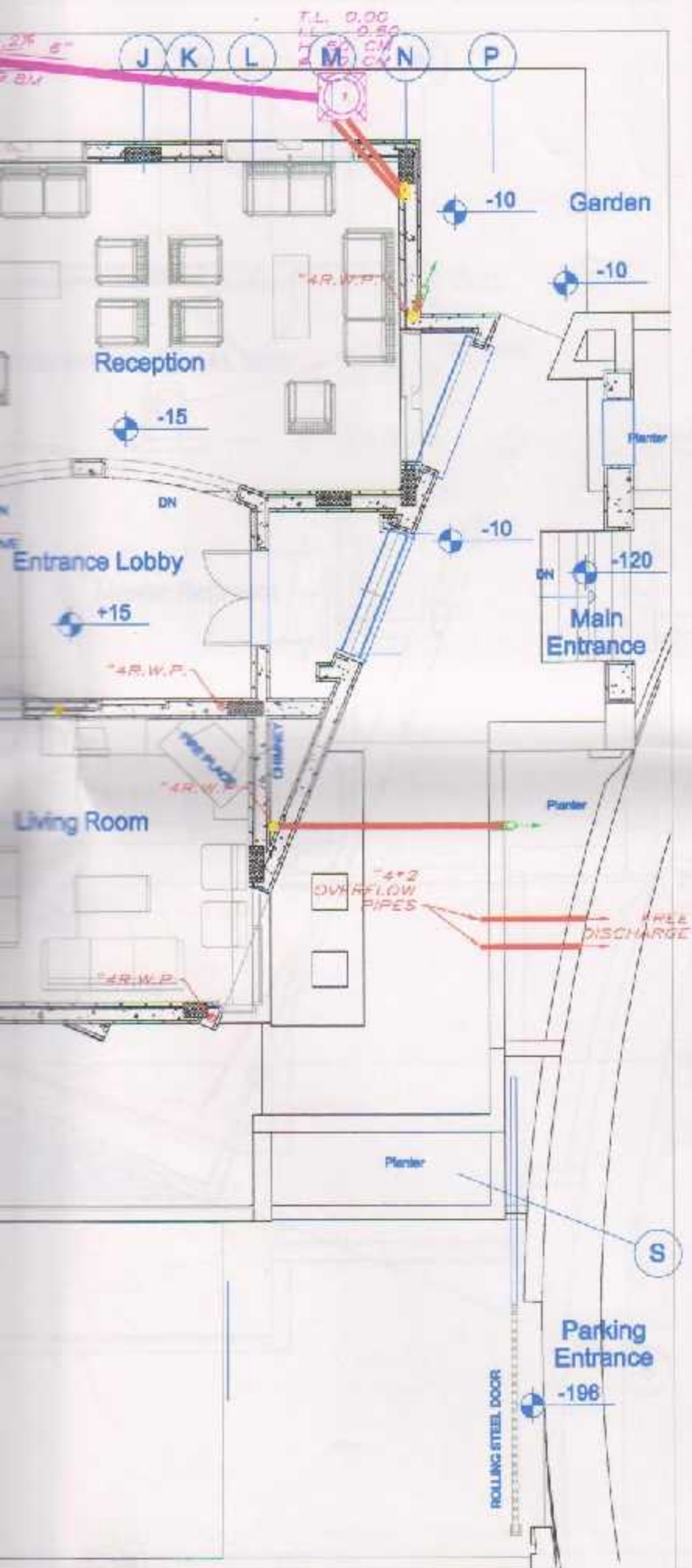
DESIGN:	DATE:
DRAWN:	
CHECKED:	
APPROVED :	SCALE: 1/100
SHEET NO: <b>M1</b>	

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SUBMERSIBLE PUMP  
10M<sup>3</sup>/HR. 3M HEAD  
1M\*1M\*1M WATER  
COLLECTION PIT



# LEGEND

- P.V.C. DRAIN PIPE  
DIAMETER (Ø) SLOPE (S)  
ماسورة تصريف بلاستيك  
قطر (Ø) ميل (S)
- 4" P.V.C. DRAIN PIPE  
(2 3/8% SLOPE)  
ماسورة تصريف بلاستيك  
قطر 4 انش ميل 2 3/8%
- 2" P.V.C. DRAIN PIPE  
(2% SLOPE)  
ماسورة تصريف بلاستيك  
قطر 2 انش ميل 2%
- 4" FLOOR TRAP  
مصيدة زوايح 4 انش
- 4 1/2" FLOOR DRAIN  
مصرف ارضي 4 1/2 انش
- 4" CLEAN OUT  
خاتمة تنظيف 4 انش
- 4" RAIN WATER PIPE  
ماسورة مطر 4 انش
- 4" RAIN WATER DRAIN  
مصرف مطر 4 انش
- 4" RAIN WATER  
(FREE OUTLET)  
مخرج حر لماسورة مطر
- 4" VENT & RISER  
ماسورة نظوية عمودية  
4 انش
- CONCRETE MANHOLE  
مناهل اسمنتي

R	REV. BY	DATE	DESCRIPTION

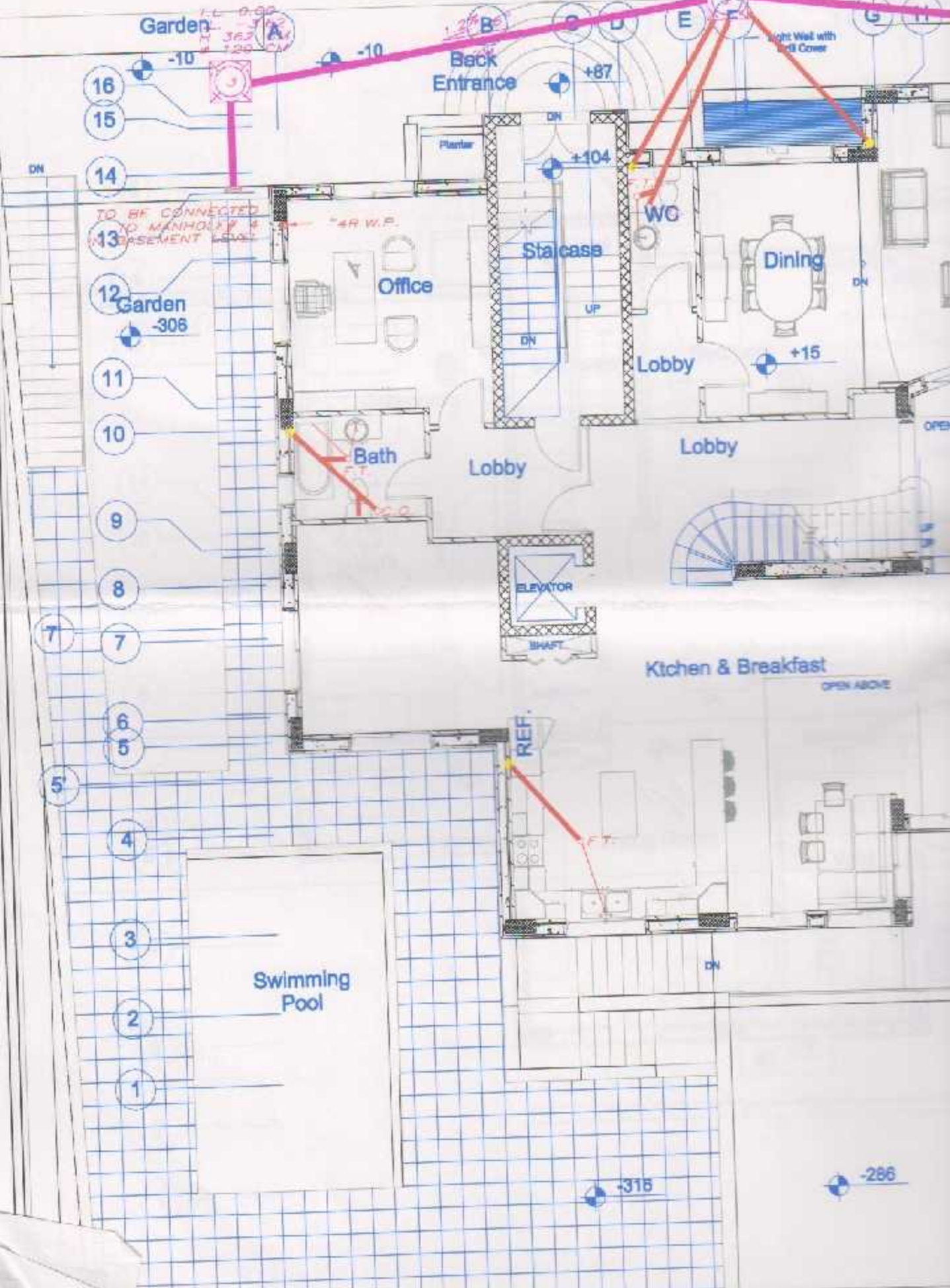
PROJECT:  
**VILLA BARHAM**  
Treh, Ramallah

DRAWING TITLE:  
  
**GROUND FLOOR  
SANITARY PLAN**

DESIGN:	DATE:
DRAWN:	
CHECKED:	
APPROVED :	SCALE:
	1/100

SHEET NO:  
**M2**

T.L. 0.00  
I.L. -0.67  
H. 67 CM  
Ø 50 CM

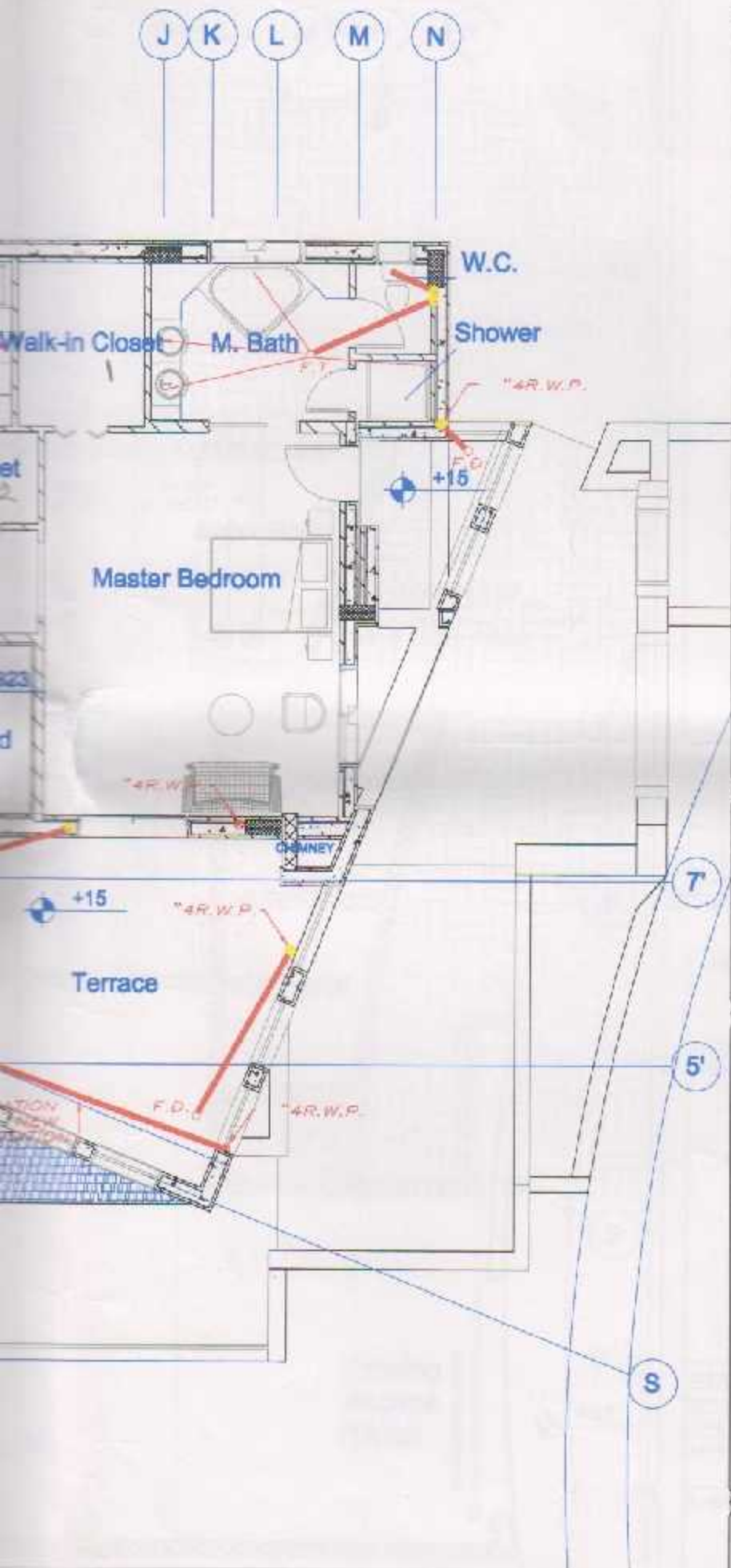


TO BE CONNECTED TO MANHOLE AT BASEMENT LEVEL

4R W.P.

Kitchen & Breakfast  
OPEN ABOVE

REF.



# LEGEND

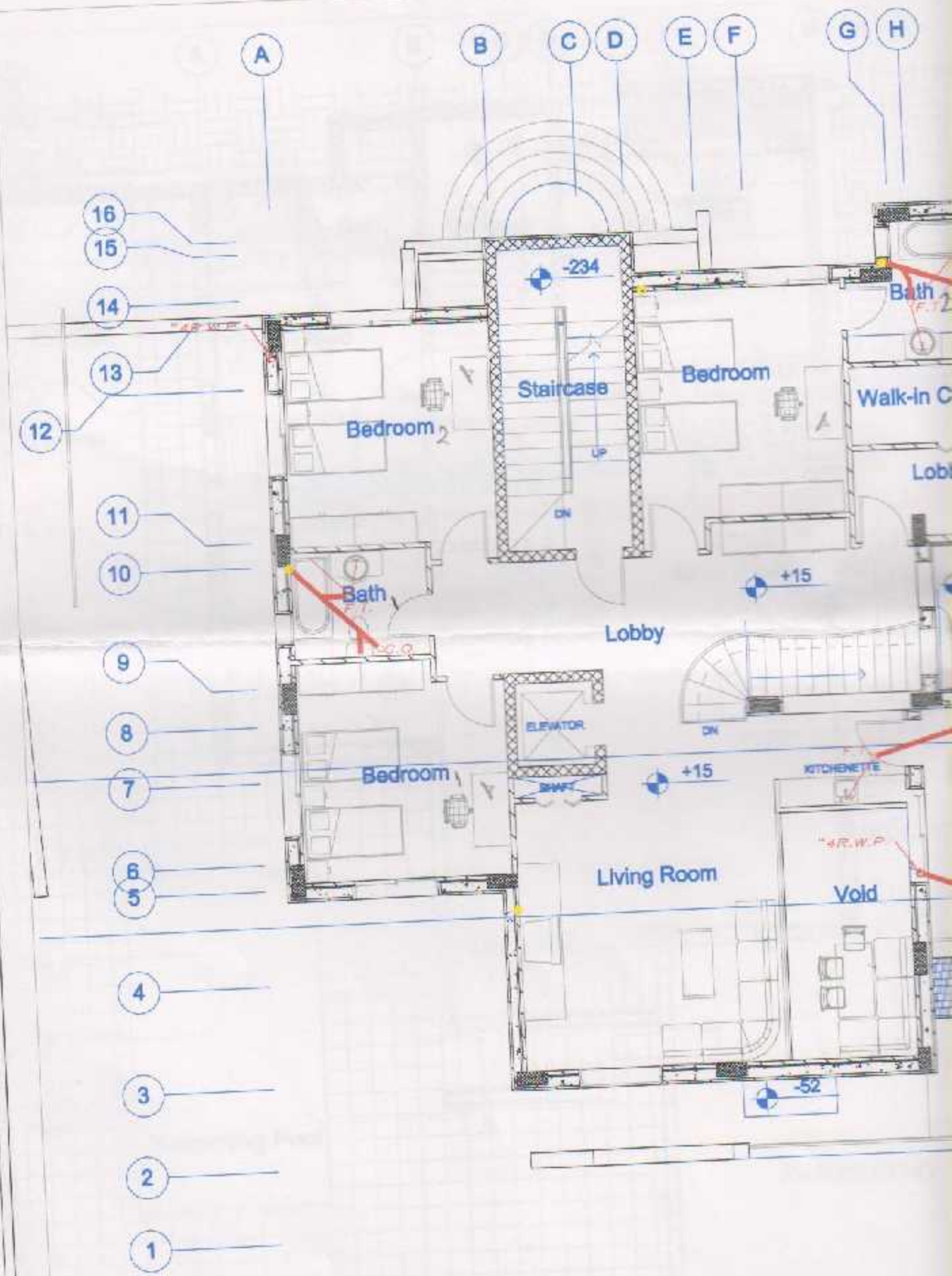
-  **P.V.C. DRAIN PIPE**  
DIAMETER (Ø) SLOPE (S)  
ماسورة تصريف بلاستيك  
قطر (Ø) ميل (S)
-  **4" P.V.C. DRAIN PIPE**  
(2% SLOPE)  
ماسورة تصريف بلاستيك  
قطر 4 انش ميل 2%
-  **2" P.V.C. DRAIN PIPE**  
(2% SLOPE)  
ماسورة تصريف بلاستيك  
قطر 2 انش ميل 2%
-  **4" FLOOR TRAP**  
مصيدة روائح 4 انش
-  **4 1/2" FLOOR DRAIN**  
صرف ارضي 4 1/2 انش
-  **4" CLEAN OUT**  
فتحة تنظيف 4 انش
-  **4" RAIN WATER PIPE**  
ماسورة مطر 4 انش
-  **4" RAIN WATER DRAIN**  
صرف مطر 4 انش
-  **4" RAIN WATER (FREE OUTLET)**  
مخرج حر لماسورة مطر
-  **4" VENT & RISER**  
ماسورة نفوية عمودية  
4 انش
-  **CONCRETE MANHOLE**  
محمل أسبكتي

R	REV. BY	DATE	DESCRIPTION

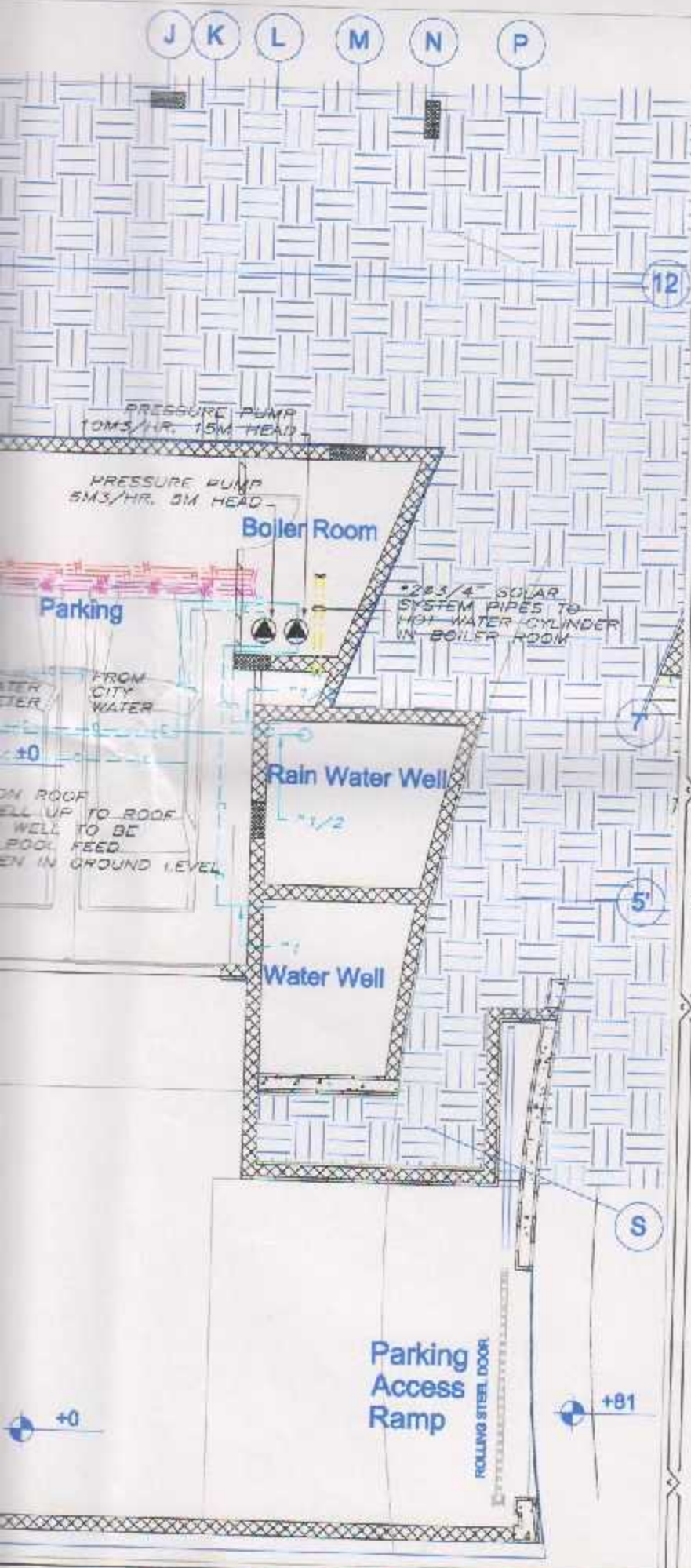
PROJECT:  
**VILLA BARHAM**  
Treh, Ramallah

DRAWING TITLE:  
**FIRST FLOOR  
SANITARY PLAN**



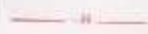





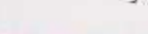
DESIGN:	DATE:
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APPROVED :	SCALE: 1/100
SHEET NO: <b>M3</b>	







# LEGEND

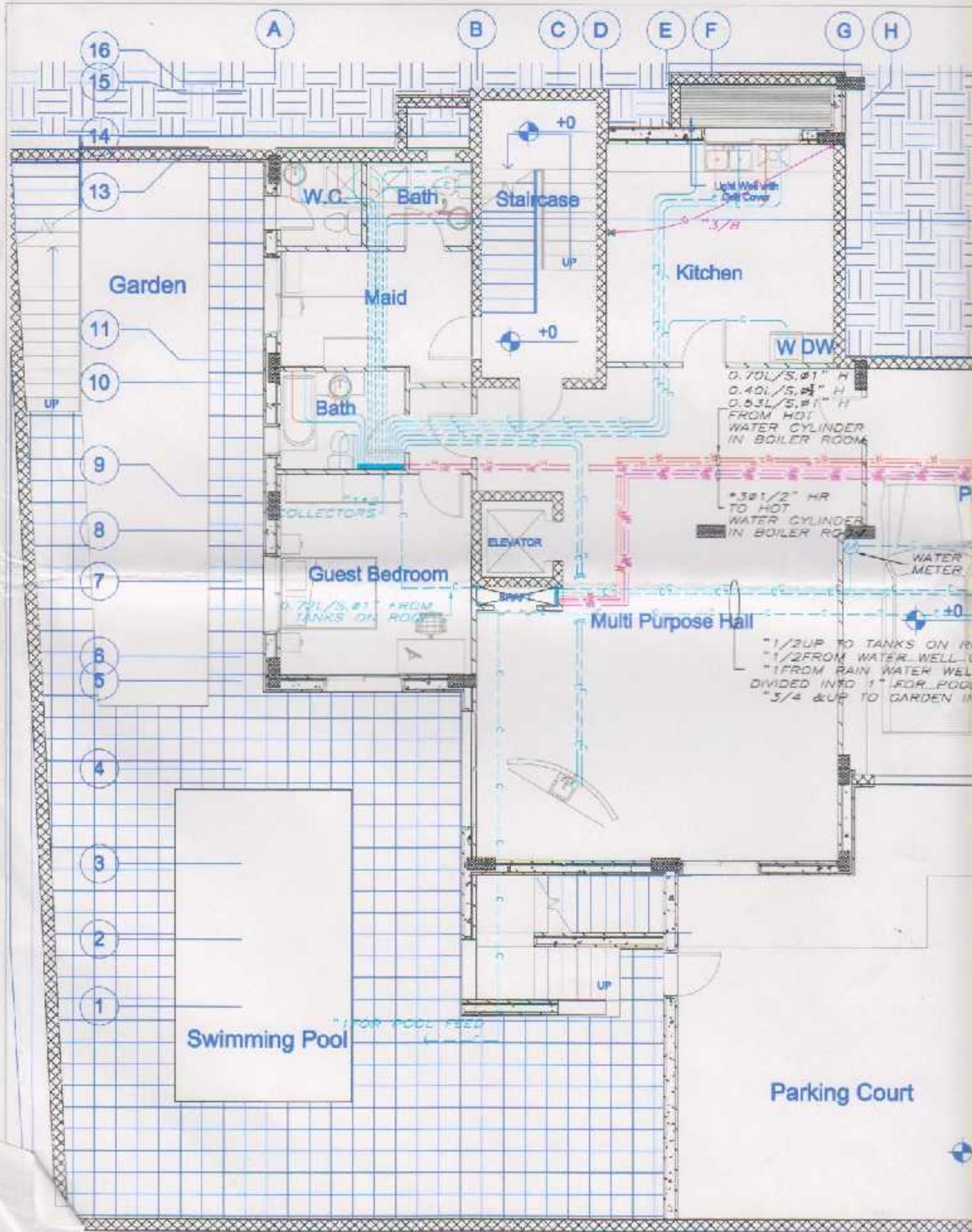
-  SHUT-OFF VALVE  
صمام اغلاق
-  FLOAT VALVE  
صواب
-  HOT WATER G.S. PIPE  
مأسورة ماء ساخن فولاذية
-  COLD WATER G.S. PIPE  
مأسورة ماء بارد فولاذية
-  HOT WATER POLYETHYLENE PIPE  
انبوب ماء ساخن من البوليثلين
-  COLD WATER POLYETHYLENE PIPE  
انبوب ماء بارد من البوليثلين
-  GAS SOFT COPPER PIPE  
انبوب نحاسي للاستعمال الغاز
-  HOT WATER RETURN PIPE  
انبوب راجع للماء الساخن
-  SOLAR SYSTEM PIPE  
مواسير النظام الشمسي

R	REV. BY	DATE	DESCRIPTION

PROJECT:  
**VILLA BARHAM**  
 Treh, Ramallah

DRAWING TITLE:  
**BASEMENT FLOOR  
 GAS & WATER PIPING PLAN**

DESIGN:	DATE:
DRAWN:	
CHECKED:	
APPROVED :	SCALE: 1/100
SHEET NO: <b>M4</b>	



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A

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D

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G

H

Garden

W.C.

Bath

Staircase

Kitchen

Bath

COLLECTORS

Guest Bedroom

ELEVATOR

Multi Purpose Hall

Swimming Pool

Parking Court

Light Well with  
Grid Cover

W.D.W.

0.70L/S, 1/2" H  
0.40L/S, 1/2" H  
0.53L/S, 1/2" H  
FROM HOT  
WATER CYLINDER  
IN BOILER ROOM

3/4" HR  
TO HOT  
WATER CYLINDER  
IN BOILER ROOM

WATER  
METER

1/2" UP TO TANKS ON ROOF  
1/2" FROM WATER WELL  
1" FROM RAIN WATER WELL  
DIVIDED INTO 1" FOR POOL  
3/4" & UP TO GARDEN

0.70L/S, 1/2" H FROM  
TANKS ON ROOF

FOR POOL FEED

+0

+0

+0

DOWN

UP

UP

For Basement floor

ROOM NAME	TOTAL LOAD IN Kcal
MULTIPURPOSE HALL	5701
GUEST ROOM	1398
BATH ROOM (1)	734
MAID ROOM	1071
BATH ROOM (2)	475
KITCHEN	1540
<b>TOTAL LOAD = 10.919 Kcal=12.70 KW</b>	

Table 3-5 loads in each room in the basement floor

RAD.NO.	NO. OF SECTION	RAD LENGTH CM	TYPE
1	11	100	AL 600
2	11	100	AL 600
3	11	100	AL 600
4	11	100	AL 600
5	11	100	AL 600
6	6	60	AL 600
7	9	84	AL 600
8	4	44	AL 600
9	12	108	AL 600
<b>TOTAL</b>	<b>86 SEC.</b>	-----	-----

Table 3-6 numbers of sections for each radiator in the basement floor

For Ground floor

ROOM NAME	TOTAL LOAD IN Kcal
RECEPTION AND DINNING	6386
LOBBY (1)	622
WATER CIRCUIT	616
OFFICE	1626
BATH ROOM	490
ROOM BESIDE ELEVATOR	1549
BREAKFAST & LIVIUNG ROOM	6429
LOBBY (2)	804
ENTARANCE LOBBY	887
	<b>TOTAL LOAD = 19.409 Kcal = 22.57 KW</b>

Table 3-7 loads in each room in the ground floor

RAD.NO.	NO. OF SECTION	RAD LENGTH CM	TYPE
1	18	156	AL 600
2	16	140	AL 600
3	16	140	AL 600
4	5	54	AL 600
5	5	54	AL 600
6	14	124	AL 600
7	4	44	AL 600
8	12	108	AL 600
9	18	156	AL 600
10	16	140	AL 600
11	16	140	AL 600
12	7	68	AL 600
13	7	68	AL 600
<b>TOTAL</b>	<b>154 SEC.</b>	-----	-----

Table 3-8 numbers of sections for each radiator in the ground floor

For First floor

ROOM NAME	TOTAL LOAD IN Kcal
BATH(1)	520
M. BATH	850
BATH (2)	650
MASTER BED ROOM	2370
WALK IN CLOSET(1)	619
LOBBY (1)	493
BED ROOM	1960
LOBBY (2)	2457
LIVING ROOM	5900
BED ROOM(1)	2151
BATH(3)	482
BED ROOM(2)	1720
WALK IN CLOSET(2)	850
<b>TOTAL LOAD = 21022 Kcal = 24.45 KW</b>	

Table 3-9 loads in each room in the first floor

RAD.NO.	NO. OF SECTION	RAD LENGTH CM	TYPE
1	4	44	AL 600
2	7	68	AL 600
3	5	54	AL 600
4	18	156	AL 600
5	5	44	AL 600
6	4	84	AL 600
7	15	84	AL 600
8	19	164	AL 600
9	14	44	AL 600
10	11	132	AL 600
11	11	124	AL 600

12	17	132	AL. 600
13	4	44	AL. 600
14	14	124	AL. 600
15	7	68	AL. 600
<b>TOTAL.</b>	<b>155 SEC.</b>	-----	-----

Table 3-10 numbers of sections for each radiator in the first floor

So, now we can calculate the heating load using this method for each floor and the result as following:-

Floor NAME	TOTAL LOAD IN KCAL
BASMENT FLOOR	10.919
GROUND FLOOR	19.409
FIRST FLOOR	21.022
	<b>TOTAL LOAD=51.35 Kcal =59.71 KW</b>

Table 3-11 total loads in each floor in the villa

### 3.9 Pick-up load

For intermittently heated buildings, or in the case of the night thermostat setback, additional heat is required for raising the temperature of air, building materials and material contents of a building to the specified indoor temperature. For temperature difference of 17°C and less, for the system to reach its design temperature in 1 hour, 20% of design heat loss is enough.

### 3.10 Heating Boiler calculations & Selection

For boiler capacity = total no. of all sections in floors × heat transfer per section

$$= 395 \text{ sec.} \times 130 (\text{Kcal/h})$$

$$=51,350 \text{ (Kcal/h)}$$

We add 10,000(Kcal/h) for hot water cylinder and 10,000(KCAL/HIR) for pool hot water supply.

$$\text{Total boiler capacity} = 51,350+10,000+10,000$$

$$= 71,350 \text{ (Kcal/h)}$$

$$\text{Boiler capacity in (kW)} = (71,350 \times 1.2) / 860$$

$$=99.56 \text{ (kW)}$$

$$\approx 100 \text{ (kW)}$$

#### From catalogue

Using Boiler of the type Fondital cast-iron boiler mod. ALOR 120(See Appendix A) of the following specifications:-

Thermal Output=124.8 KW

Maximum thermal power =114.0KW



Fig3.2 Boiler

2- shower is a fixture with both cold and hot water supplies, so, the weights for maximum separate demands may be taken as three-quarters (3/4) the list demand for the supply in Table (4.1), so from Table (4.1) the shower (private) gives 2 WSFU for the total demand (for both cold and hot water ) so, for cold water only or hot water only we take ( $3/4 * WSFU = 3/4 * 2 = 1.5$ ) & ( WSFU =2 for both cold and hot water)

3- Water closet: is a fixture with cold water supply only, so, the weights for maximum separate demands may be taken as three-quarters (3/4) the list demand for the supply in Table (4.1),so from Table(4.1)the water closet (private and with flush tank) gives 3 WSFU for the total demand (for both cold and hot water ) so, for cold water only or hot water only we take ( $3/4 * WSFU = 3/4 * 3 = 2.25$ ) & ( WSFU =3 for both cold and hot water)

4- So, from the above information we can do the following table:-

Fixture Unit	No. of Units	WSFU From table(9_3)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot & Cold Water
Lavatory (private)	1	$3/4 * 1$	3/4	3/4	1
Shower (private)	1	$3/4 * 2$	1.5	1.5	2
Water Closet flush tank(private)	1	$3/4 * 3$	2.25	-----	3
-----	-----	-----	$\Sigma = 4.5$ WSFU	$\Sigma = 2.25$ WSFU	$\Sigma = 6$ WSFU

Table 4-1 WSFU for the Bath Room



Now we can calculate the number of fixture in each floor from Mechanical Drawing and so we can do the following table:-

Floor name	Total Lavatory	Total sink	Total Bath tube	Total Shower head	Total Water closet /flush tank
Basement	3	1	1	2	3
Ground floor	2	1	1	-----	2
First floor	4	1	3	1	3
Continuous flow at garden	3/4"	-----	-----	-----	-----

Table 4-2 Total number for water supply fixture unit for cold and hot water

Now we can calculate the (WSFU) for each floor, So (WSFU) for each floor as in the following tables:

Fixture Unit	No. of Units	WSFU From table (4.1)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot Water Cold Water &
<b>BASMENT FLOOR</b>	-----	-----	-----	-----	-----
Lavatory (private)	3	$\frac{3}{4} * 1$	2.25	2.25	3
Kitchen Sink(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Bath tube(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Water Closet flush tank(private)	3	$\frac{3}{4} * 3$	6.75	-----	9
Continuous flow for Garden	1	$\frac{3}{4} * 3$	2.25	-----	3
-----	-----	-----	$\Sigma = 14.25$ WSFU	$\Sigma = 5.25$ WSFU	$\Sigma = 18$ WSFU

Table 4-3 Water supply unit for cold and hot water for basement floor

Fixture Unit	No. of Units	WSFU From table (4.1)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot Water Cold Water &
<b>GROUND FLOOR</b>	-----	-----	-----	-----	-----
Lavatory (private)	2	$\frac{3}{4} * 1$	1.5	1.5	2
Kitchen Sink(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Bath tube(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Water Closet flush tank(private)	2	$\frac{3}{4} * 3$	4.5	-----	6
-----	-----	-----	$\Sigma = 9$ WSFU	$\Sigma = 4.5$ WSFU	$\Sigma = 12$ WSFU

Table 4-4 Water supply unit for cold and hot water for ground floor

Fixture Unit	No. of Units	WSFU From table (9.3)	Total no. of WSFU For Cold Water	Total no. of WSFU For hot Water	Total no. of WSFU For hot Water Cold Water &
<b>FIRST FLOOR</b>	-----	-----	-----	-----	-----
Lavatory (private)	4	$\frac{3}{4} * 1$	3	3	4
Kitchen Sink(private)	1	$\frac{3}{4} * 2$	1.5	1.5	2
Bath tube(private)	3	$\frac{3}{4} * 2$	4.5	4.5	6
Water Closet flush tank(private)	3	$\frac{3}{4} * 3$	6.75	-----	9
-----	-----	-----	$\Sigma = 15.75$ WSFU	$\Sigma = 9$ WSFU	$\Sigma = 21$ WSFU

Table 4-5 Water supply units for cold and hot water for first floor

#### 4.2.2 Flow rate calculations

To calculate flow rate in gpm:-

##### FOR BASMENT FLOOR

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank for basement floor the estimating demand in gpm for cold water = 14.25 WSFU.

So by using interpolation = 10.55 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for basement floor estimating demand in gpm for hot water = 5.25 WSFU.

So by using interpolation = 4.25 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for basement floor estimating demand in gpm for hot & cold water = 18 WSFU.

So by using interpolation = 12.80gpm

**Note:** - For supply outlets likely to impose continuous demand, we estimate continuous separately and add to the total demand for fixture

We have 1 faucets (Ø 3/4) for garden so from Table 4-2(See Appendix B)

The total flow in gpm = 2.25 WSFU. = 2 gpm

So the total cold and hot water demand needed = 12.80 + 2 = 14.80 gpm

### FOR GROUND FLOOR

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank for ground floor the estimating demand in gpm for cold water = 9 WSFU.

So by using interpolation = 7.25 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for ground floor estimating demand in gpm for hot water = 4.5 WSFU.

So by using interpolation = 3.875 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for ground floor estimating demand in gpm for hot & cold water = 12 WSFU.

So by using interpolation = 9.2 gpm

So the total cold and hot water demand needed = 12 WSFU. = 9.2 gpm

### FOR FIRST FLOOR

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank for first floor the estimating demand in gpm for cold water = 15.75 WSFU.

So by using interpolation = 11.45 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for first floor estimating demand in gpm for hot water = 9 WSFU.

So by using interpolation = 7.25 gpm

By using Table 4-2 (See Appendix B) for supply system predominantly for flush tank the for first floor estimating demand in gpm for hot & cold water = 21 WSFU.

So by using interpolation = 14.6 gpm

So the total cold and hot water demand needed = 14.6 gpm

**To calculate static head for Basement floor for cold water:-**

Floor to floor height = 4.00

We have 3 floor (basement + ground + first) and so  $4.0 \times 3.0 = 12.0$

Sink outlet above basement level = 1.05 m

& tank outlet above roof level = 0.75

So there is static head in this case =  $12 + 0.75 - 1.05 = 11.40 \text{ m} = 16.22 \text{ psi}$

**To calculate the equivalent length:-**

Pumps at Basement floor transfer the water through pipes to the tank at Roof from which it distribute the water through pipes to the collectors at the First, Ground & Basement floor

So we will calculate the equivalent length from the tank at roof to the farthest outlet at the Basement floor (sink) at farthest collector

#### **FOR COLD WATER**

Total length through shaft room from Roof to the Basement floor = 16.0 m

Total length in the Basement floor from riser at the shaft room to farthest out let (sink) at farthest collector = 18.0 m